Chains & sprockets	Wear strips	Side guide brackets & accessories	Levelers	Engineering manual
Modular belts & sprockets	Chain & belt return systems	Frame & structure supports	Bearing supports	
Curves & tracks	Side guide solutions	Miscellaneous products	Equipment	





GENERAL INDEX









LEGEND ICONS



ICON	LEGEND
ALL STORY OF THE S	Link to open menu for technical information related to the products listed on this page.
1 1 0 1 1 a	Link to find related products.
В 111150 Б	Link to find related products. Note: only available on our website www.SystemPlastSmartGuide.com
3 0 7 2 0 a	Codes in orange background are indicators for modular belts in molded to width versions.
	Information only available on our website: www.SystemPlastSmartGuide.com
	Link to other varieties of these chains and/ or link to the chains used with the sprockets listed on this page.
32222	Link to other varieties of these modular belts and/or link to the belts used with the sprockets listed on this page.
•	Link to the sprockets used with the chains or belts listed on this page.
MOQ	Minimum order quantity.
Ů	Indicates chains, belts or curves suitable for magnetic curve systems.
PATENTED Patented Feature	This product includes patented features.
(i)	Link to general information about the products listed on this page.
	This product is not suitable in use with chains with hold-down tabs.

ICON	LEGEND			
Kg/m	Weight in Kg/m			
4	Length per standard code.			
	Length per standard coil.			
	Quantity standard packaging.			
(5)	Indication for products that contain a dry lubricant.			
SS 430	Material: stainless steel AISI 430.			
SS 304	Material: stainless steel AISI 304.			
Al-a	Material: aluminum, anodized.			
A	Material: aluminum.			
Zn	Material: zinc plated steel.			
Ø 41	Nominal diameter 41mm.			
	Product suitable for assembly to a rectangular profile.			
	Max load capacity.			
>	Wrench size.			

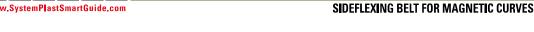




12,7 mm (1/2") MODULAR BELTS 2120M









Reduce noise level by 12 dB

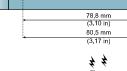


12,7 mm



RD=15 mm

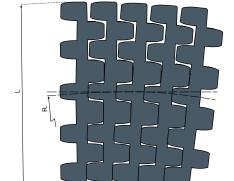
(RD=0.59 in)





K330 - 83,8 mm

(K330 - 3.3 in)



TRAVEL

Features:

More.

This system offers a unique solution for sideflexing applications in plants with modular conveyors.

Applications:

- To be used in the dry end of the line where bottles or cans run in lanes separated by guides.
- The short pitch makes in-line transfers possible.

Weight: 13,7 kg/m² (2.8 lbs/ft².) Pin material: Ferritic Stainless Steel

stated by EC1935/2004 and FDA CFR21 norms for direct food contact Less dust than with acetal chains

ITEM DESCRIPTION	MATERIAL	WIDTH L		R	MAX LOAD CAPACITY		WEIGHT	
ITEM DESCRIPTION	WAILNIAL	mm	in	MIN.	N	lbs	Kg/m	lbs/ft.
NGE2120M-SFF-K330	NGE Grey Blue	83,8	3.30	500	1900	420	1,1	0.74

Standard length: 240 pitches (3,048 m - 10 ft.)



Information about this product is only available on our Smart Guide® website.

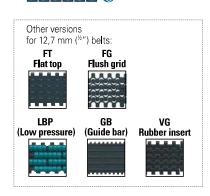
Standard material

NGE (GREY BLUE)

New Generation®

materials details:

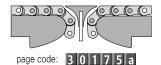








Can be used with a Nose bar to create extremely short transfers.



Chains & sprockets	Wear strips	Side guide brackets & accessories	Levelers	Engineering manual
Modular belts & sprockets	Chain & belt return systems	Frame & structure supports	Bearing supports	
Curves & tracks	Side guide solutions	Miscellaneous products	Equipment	





APPLICATION EXAMPLES





HEAVY DUTY BELTS IN CAR MANUFACTURING.

Go to:

3 0 6 2 0 a

Go to:

3 0 6 4 5 a





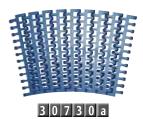




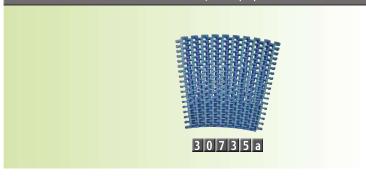


INDEX SIDEFLEXING MODULAR BELTS

SIDEFLEXING BELT 2256 - PITCH: 25,4 mm (1")



SIDEFLEXING BELT 2256 - PITCH: 25,4 mm (1") VERSION WITH RUBBER INSERTS

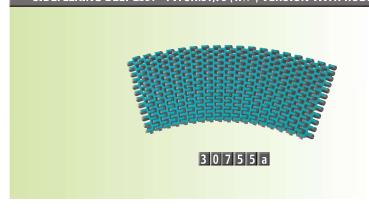


SIDEFLEXING BELT 2351/2451/2551/2651 - PITCH: 31,75 mm (1.1/4")



CURVE RATIO	STANDARD	HEAVY DUTY
1,6	3 0 7 4 5 a	3 0 7 5 0 a
1,0	5 U / 4 5 d	5 U 7 5 U d

SIDEFLEXING BELT 2351 - PITCH:31,75 (1.1/4") VERSION WITH RUBBER INSERTS

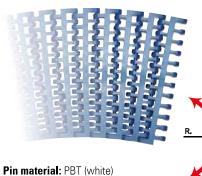




25,4 mm (1") MODULAR BELTS 2256







152,4 mm 12,7 mm (0.5 in) 6,35 mm (0.25 in)



Other versions for 25,4 mm (1") belts: VG Rubber insert

Open surface: 29% (straight) Standard length:

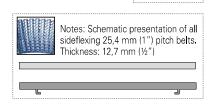
width ≤ 24 in: 3,048 m (10 ft.) width > 24 in: 1,524 m (5 ft.)



	٥.		
*	Sing	е	track

WIDTH W		SIDE FLEX	MAX LOAD CAPACITY			BACK FLEX RADIUS		WEIGHT		
mm	ı in	R. MIN.	STRA	AIGHT CURVE			1	1/ / 2 Ib //4		
			N/m	lbs/ft.	N	lbs	mm	in	Kg/m²	lbs/ft.
76*	3	145	20000	2000	2000	2000 450	25	0.98	7,75	1.58
152*	6	290								
229	9	435								
305	12	580								
381	15	725	30000							
457	18	890								
533	21	1040								
610	24	1250								

This belt is bi-directional and can make both left and right turns. Even S-curves are possible if the load allows for that.



NG® evo conveyor components made from engineered plastic resin

Standard materials

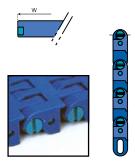
LFB (BLUE)	NGE (GREY BLUE)
Low friction acetal resin	New Generation®

materials details:

7 3 5 6 3 a 🗎

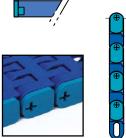
Without TAB "C" version

The belt is available with several clips and guiding systems.



Just a clip for pin retention.

With slider shoe "S" version

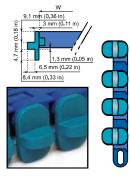


Attached to the pin retention clip is a slider shoe that can reduce friction and act as a wear part.

With TAB "T" version mm (0.33 in) 3 mm (0.11 in) 6,5 mm (0. 5,6 mm (0.22 in) 1,3 mm (0.05 in)

Attached to the pin retention clip is a TAB that holds the belt down in the curves.

With TAB & positioner "TP" version



Attached to the pin retention clip is a TAB-Positioner combination that holds the belt down in the curves but also guides the belt on straight sections.

ITEM DESCRIPTION
LFB2256TP-K300
LFB2256TP-K600
LFB2256TP-K900
LFB2256TP-K1200
LFB2256TP-K1500
LFB2256TP-K1800
LFB2256TP-K2100
LFB2256TP-K2400

* Single track

onigic track					
TH W					
in					
3					
6					
9					
12					
15					
18					
21					
24					

VIDT	TH W	ITEM DESCRIPTION
m	in	ITEM DESCRIPTION
*6	3	LFB2256C-K300
2*	6	LFB2256C-K600
29	9	LFB2256C-K900
)5	12	LFB2256C-K1200
31	15	LFB2256C-K1500
57	18	LFB2256C-K1800
33	21	LFB2256C-K2100
10	24	LFB2256C-K2400

ITEM DESCRIPTION LFB2256S-K300 LFB2256S-K600 LFB2256S-K900 LFB2256S-K1200 LFB2256S-K1500 LFB2256S-K1800 LFB2256S-K2100 LFB2256S-K2400

ITEM DESCRIPTION LFB2256TAB-K300 LFB2256TAB-K600 LFB2256TAB-K900 LFB2256TAB-K1200 LFB2256TAB-K1500 LFB2256TAB-K1800

LFB2256TAB-K2100

LFB2256TAB-K2400

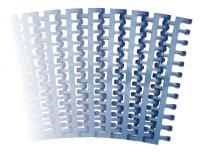




25,4 mm (1") MODULAR BELTS 2256ST TAB



SIDEFLEXING

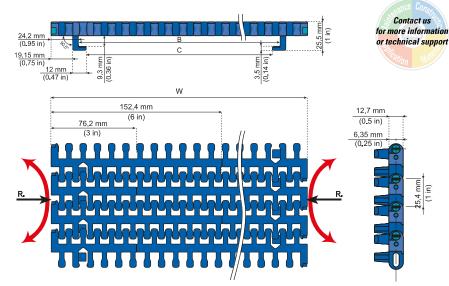


Pin material: PBT (white)

Available Widths: 4" single track, 6" single track, 6" + multiple of 3" (9", 12", 15", ...)

Opening Size (approximate): 8.5x73 mm (0.33"x0.28") Open Area: 29% (once straight) Drive Method: Hinge-driven Materials: POM e NGE

Contact System Plast for precise belt measurements and stock status before designing equipment or ordering a belt.



WIDT	TH W	ı	3	(C	R N	IIN.	MAX LOAD CAP		DAD CAPACITY		BACK FLE	X RADIUS	WE			
mm	in	mm	in	mm	in	mm	in	STRAI		CUI		mm	in	Kg/m²	lbs/ft.		
101,6*	4	53,1	2.09	39,3	1.55	250	9.84	N/m	lbs/ft.	N	lbs	25	0.00	775	1.50		
152,4*	6	103,9	4.09	90,1	3.55	335	13.19	30000	2000	2000	450	25	0.98	7,75	1.58		
228,6	9	180,1	7.09	166,3	6.55	500	19.69										
304,8	12	256,3	10.09	242,5	9.55	690	27.17										
381	15	332,5	13.09	318,7	12.55	870	34.25	30000	30000	+ 30000	2000	2000	450	25	0.98	7.75	1.58
457,2	18	408,7	16.09	394,9	15.55	1050	41.34				2000	2000	450	25	0.90	7,75	1.36
533,4	21	484,9	19.09	471,1	18.55	1225	48.23										
609,6	24	561,1	22.09	547,3	21.55	1400	55.12										

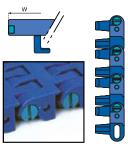
This belt is bi-directional and can make ooth left and right turns. Even S-curves are possible if the load allows for that. The belt is available with several clips and guiding systems.

> Other versions for 25,4 mm (1") belts:

Rubber insert

* Single track

Clips without TAB 'C" Version



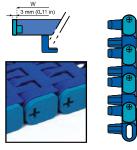
Just a clip for pin retention.

* Single track

WIDTH W							
mm	in						
101,6*	4						
152,4*	6						
228,6	9						
304,8	12						
381	15						
457,2	18						
533,4	21						
609,6	24						

ITEM DESCRIPTION LFB2256ST-C-K400 LFB2256ST-C-K600 LFB2256ST-C-K900 LFB2256ST-C-K1200 LFB2256ST-C-K1500 LFB2256ST-C-K1800 LFB2256ST-C-K2100 LFB2256ST-C-K2400

Clips with slider shoe "S" version



Attached to the pin retention clip is a slider shoe that can reduce friction and act as a wear part.

ITEM DESCRIPTION
LFB2256ST-S-K400
LFB2256ST-S-K600
LFB2256ST-S-K900
LFB2256ST-S-K1200
LFB2256ST-S-K1500
LFB2256ST-S-K1800
LFB2256ST-S-K2100
LFB2256ST-S-K2400







NG[®]*evo* conveyor components made from engineered plastic resin..

Standard materials

LFB (BLUE)	NGE (GREY BLUE)
Low friction acetal resin	New Generation®

materials details:





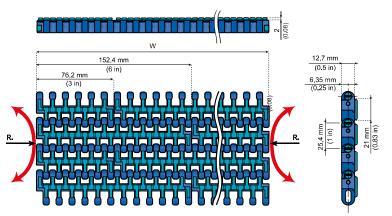


25,4 mm (1") MODULAR BELTS 2256 - RUBBER INSERT



SIDEFLEXING









Belt material: low friction acetal resin, dark blue colour

Rubber material: thermoplastic rubber, waterblue colour, 75 shore A.

Pin material: PBT (white); open surface: 29% (straight)

Standard length: width ≤ 24 in: 3,048 m (10 ft.) / width > 24 in: 1,524 m (5 ft.)

WID	TH W	SIDE FLEX	MAX LOAD (CAPACITY	ВАСК	
mm	in	RADIUS MIN.			FLEX RADIUS	WEIGHT
76*	3	145				
152*	6	290				
229	9	435				
305	12	580	30000 N/m	2000 N	25 mm	7,75 Kg/m²
381	15	725	(2000 lbs/ft.)	(450 l bs)	(0.98 in)	(1.58 lbs/ft ² .)
457	18	890				
533	21	1040				
610	24	1250				



materials details:

7 3 5 6 3 a 🕀







Notes: Schematic presentation of

belts. Thickness: 12,7 mm (1/2")

all sideflexing VG 25,4 mm (1") pitch

This belt is bi-directional and can make both left and right turns. Even S-curves are possible if the load allows for that. The belt is available with several clips and guiding systems.

With TAB

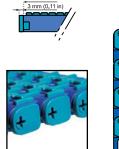
Without TAB "C" version





Just a clip for pin retention.

With slider shoe "S" version

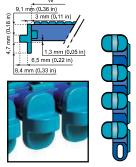


Attached to the pin retention clip is a slider shoe that can reduce friction and act as a wear part.

"T" version 6,5 mm (0.22 in)

Attached to the pin retention clip is a TAB that holds the belt down in the curves.

With TAB & Positioner "TP" version



Attached to the pin retention clip is a TAB-Positioner combination that holds the belt down in the curves but also guides the belt on straight sections.

*	Single	track

ITEM DESCRIPTION	WIDTH W		
	in	ım	
LFB2256C-K300	3	6*	
LFB2256C-K600	6	52*	
LFB2256C-K900	9	29	
LFB2256C-K1200	12	05	
LFB2256C-K1500	15	81	
LFB2256C-K1800	18	57	
LFB2256C-K2100	21	33	
LFB2256C-K2400	24	10	

ITEM DESCRIPTION LFB2256S-K300VG LFB2256S-K600VG LFB2256S-K900VG LFB2256S-K1200VG LFB2256S-K1500VG LFB2256S-K1800VG LFB2256S-K2100VG LFB2256S-K2400VG

ITEM DESCRIPTION LFB2256TAB-K300VG LFB2256TAB-K600VG LFB2256TAB-K900VG LFB2256TAB-K1200VG LFB2256TAB-K1500VG LFB2256TAB-K1800VG

LFB2256TAB-K2100VG

LFB2256TAB-K2400VG

ITEM DESCRIPTION
LFB2256TP-K300VG
LFB2256TP-K600VG
LFB2256TP-K900VG
LFB2256TP-K1200VG
LFB2256TP-K1500VG
LFB2256TP-K1800VG
LFB2256TP-K2100VG
LFB2256TP-K2400VG

^{*} Single track





25,4 mm (1") MODULAR BELTS 2256ST VG TAB - RUBBER INSERT

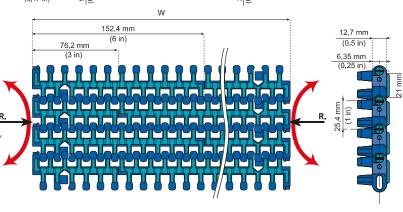












Pin material: PBT (white)

Available Widths: 4" single track, 6" single track,

6" + multiple of 3" (9", 12", 15", ...) Opening Size (approximate): 8.5x73 mm (0.33"x0.28") **Open Area:** 29% (once straight) Drive Method: Hinge-driven Materials: POM e NGE

Rubber material: thermoplastic rubber, waterblue colour, 75 shore A.

Contact System Plast for precise belt measurements and stock status before designing equipment or ordering a belt.

1<u>9,15 mm</u> (0.75 in)

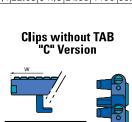
WIDT	TH W	ı	3	(C	R N	IIN.	MAX LOAD CA		X LOAD CAPACITY		BACK FLE	X RADIUS	WEIGHT	
mm	in	mm	in	mm	in	mm	in	STRAI		CUI		mm	in	Kg/m²	lbs/ft.
								N/m	lbs/ft.	N	bs				120,14
101,6*	4	53,1	2.09	39,3	1.55	250	9.84	30000	2000	2000	450	25	0.98	7.75	1.58
152,4*	6	103,9	4.09	90,1	3.55	335	13.19	30000	2000	2000	450	20	0.30	1,75	1.00
228,6	9	180,1	7.09	166,3	6.55	500	19.69								
304,8	12	256,3	10.09	242,5	9.55	690	27.17								
381	15	332,5	13.09	318,7	12.55	870	34.25	30000	2000	2000	450	25	0.98	7.75	1.58
457,2	18	408,7	16.09	394,9	15.55	1050	41.34	30000	2000	2000	450	25	0.98	1,75	1.58
533,4	21	484,9	19.09	471,1	18.55	1225	48.23								
609,6	24	561,1	22.09	547,3	21.55	1400	55.12								

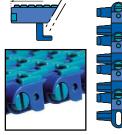
This belt is bi-directional and can make both left and right turns. Even S-curves are possible if the load allows for that. The belt is available with several clips and guiding systems.

> Other versions for 25,4 mm (1") belts:

> > FG Flush Grid

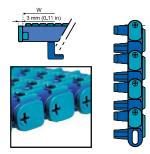
* Single track





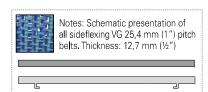
Just a clip for pin retention.

Clips with slider shoe "S" version



Attached to the pin retention clip is a slider shoe that can reduce friction and act as a wear part.

ITEM DESCRIPTION
LFB2256ST-S-K400VG
LFB2256ST-S-K600VG
LFB2256ST-S-K900VG
LFB2256ST-S-K1200VG
LFB2256ST-S-K1500VG
LFB2256ST-S-K1800VG
LFB2256ST-S-K2100VG
LFB2256ST-S-K2400VG



3 0 7 4 0 a

NG[®]*evo* conveyor components made from engineered plastic resin..

Standard materials

LFB (BLUE)	NGE (GREY BLUI
Low friction acetal resin	New Generation

materials details:



* Single track

WIDTH	ł W
mm	in
101,6*	4
152,4*	6
228,6	9
304,8	12
381	15
457,2	18
533,4	21
609,6	24

	ITEM DESCRIPTION
	LFB2256ST-C-K400VG
	LFB2256ST-C-K600VG
r	LFB2256ST-C-K900VG
	LFB2256ST-C-K1200VG
	LFB2256ST-C-K1500VG
	LFB2256ST-C-K1800VG
	LFB2256ST-C-K2100VG
	LFB2256ST-C-K2400VG





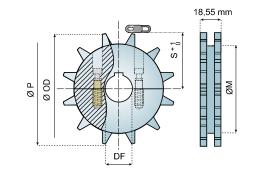
DRIVE AND RETURN SPROCKETS





Features:

- Very high strength.
- Split versions: quick and easy replacement.
- Foolproof design.
- Completely closed structure, easy to clean and to disinfect.
- Excellent wear and chemical resistance.
- Recyclable.





FOR BELT 2256 (VG)

z	Ø P mm	Ø OD mm	Ø M mm	S mm
10	82,2	83,3	65,3	33
12	98,1	100	82	41
15	122,2	124,3	106,3	53
18	146,3	147,8	129	65

MACHINED DRIVE SPROCKETS - SPLIT										
Z	Ø 25	Ø 30	Ø 40							
2		ITEM DESCRIPTION								
10	2256-10R25M-DPMS	2256-10R30M-DPMS	-							
12	2256-12R25M-DPMS	2256-12R30M-DPMS	2256-12R40M-DPMS							
15	2256-15R25M-DPMS	2256-15R30M-DPMS	2256-15R40M-DPMS							
18	2256-18R25M-DPMS	2256-18R30M-DPMS	2256-18R40M-DPMS							
MACHINED RETURN SPROCKETS - SPLIT										
10	2256-10R25M-RMS	2256-10R30M-RMS	-							
12	2256-12R25M-RMS	2256-12R30M-RMS	2256-12R40M-RMS							
15	2256-15R25M-RMS	2256-15R30M-RMS	2256-15R40M-RMS							
18	2256-18R25M-RMS	2256-18R30M-RMS	2256-18R40M-RMS							
	MACHINED DRIVE SPROCKETS FLOATING	SQUARE BORE - SPLIT								
Z	Ø 25X25	⊠ 30X30	Ø 40X40							
10	-	2256-10S30M-DMS	-							
12	2256-12S25M-DMS	2256-12S30M-DMS	-							
15	<u>-</u>	2256-15S30M-DMS	2256-15S40M-DMS							
18	2256-18S25M-DMS	-	2256-18S40M-DMS							
ateri	al: polyamide (natural white), scre	ws in stainless steel, bushings in	brass.							









APPLICATION EXAMPLES





SPEEDRAIL ™ ROLLER GUIDES FOR SHRINK WRAPPED PRODUCTS.

Go to:

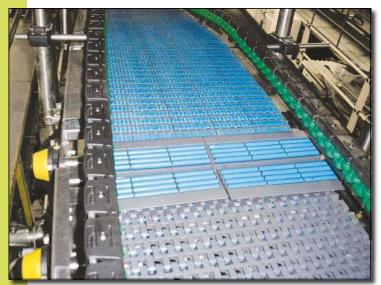
5 2 5 1 0 a



LUGGAGE HANDLING BELT WITH MODULAR BELTS WITH RUBBER INSERTS.

Go to:

3 0 4 9 0 a



ROLLER TRANSFER MODULES.

Go to:

5 2 6 7 0 a



31,75 mm (1 1/4") MODULAR BELTS 2351 & 2551



SIDEFLEXING

31,75 mm (1.25 in)

Features for 2351:

- Radius width ratio min. 1,6
- Can also be used in S-curves Weight: 9.0 kg/m² (1.84 lbs/f².)



Features for 2551:

• Radius – width ratio min. 1,0 for tight curves Weight: 8.2 kg/m² (1.67 lbs/f².)

Features for both:

- Optimum product support
- Strong design
- For left or right L- or U-turns
- Easy installation and maintenance

Pin Material: PBT white Standard length:

width ≤ 850 mm (33.46 in): 1,524 m (5 ft.) width > 850 mm (33.46 in): on request



Information about this product is only available on our Smart Guide® website.

Standard material

LFG (GREY)

Low friction acetal resin

materials details:



Also available into

LFW (COLOR WHITE)

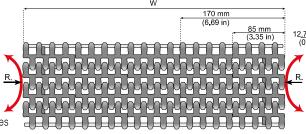
Low friction acetal resin

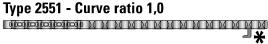
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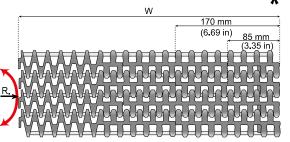
materials details:





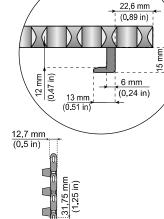






1450

850



(0.59 in)

Ÿ												
		2351			255	1	MAX LOAD	CAPACITY	BACK			
	WID:	WIDTH W SIDE FLEX		WID	WIDTH W SIDE FLEX		STRAIGHT	STRAIGHT CURVE				
	mm	in	R. MIN.	mm	in	R. MIN.	STRAIGHT	CONVE	RADIUS			
	255	10.04	408	-	-	-						
	340	13.39	545	340	13.39	340						
	425	16.73	680	425	16.73	425						
	510	20.08	840	510	20.08	510	30000 N/m	2500 N	25 mm			
	595	23.43	980	595	23.43	595	(2000 lbs/ft.)	(560 l bs)	(0.98 in)			
	680	26.77	1150	680	26.77	680						
	765	30.12	1300	765	30.12	850						

Side flex radius values are valid for curves up to 180 degrees. For larger angles or for spirals, please consult our Application Engineering.

1050

33.46

2351



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PATENTED

3 0 7 6 0 a

WID.	TH W	ITEM DESCRIPTION	ITEM DESCRIPTION
mm	in	I I EINI DESCRIFTION	I LEINI DESCRIPTION
255	10.04	LFG2351=M0255	LFG2351TAB=M0255
340	13.39	LFG2351-M0340	LFG2351TAB-M0340
425	16.73	LFG2351-M0425	LFG2351TAB-M0425
510	20.08	LFG2351-M0510	LFG2351TAB-M0510
595	23.43	LFG2351-M0595	LFG2351TAB-M0595
680	26.77	LFG2351-M0680	LFG2351TAB-M0680
765	30.12	LFG2351-M0765	LFG2351TAB-M0765
850	33.46	LFG2351-M0850	LFG2351TAB-M0850



Without TAB With TAB 1 side

WID.	TH W	ITEM DESCRIPTION	ITEM DESCRIPTION
mm	in	I I EINI DESCRIFTION	I I EIVI DESCRIFTION
-	-	-	-
340	13.39	LFG2551-M0340	LFG2551TAB-M0340
425	16.73	LFG2551-M0425	LFG2551TAB-M0425
510	20.08	LFG2551-M0510	LFG2551TAB-M0510
595	23.43	LFG2551-M0595	LFG2551TAB-M0595
680	26.77	LFG2551-M0680	LFG2551TAB-M0680
765	30.12	LFG2551-M0765	LFG2551TAB-M0765
850	33.46	LFG2551-M0850	LFG2551TAB-M0850





31,75 mm (1 1/4") MODULAR BELTS 2451 & 2651

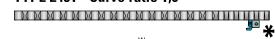


• Radius - width ratio min. 1,6 Weight: 9.7 kg/m² (1.98 lbs/ft².)

Features for 2451:

SIDEFLEXING HEAVY DUTY

TYPE 2451 - Curve ratio 1,6





*

<u>=</u>

(0.59 i

19 mm

(0.75 in)

(0.47 in)

(0.65 in) 19,5 mm

(0.77 in)

16,5 mm

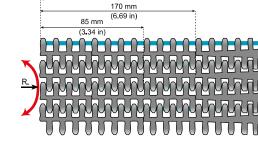
12,7 mm

(0.5 in)

12,7 mm

(0.5 in)





TYPE 2651 - Curve ratio 1.0

255 mm 170 mm (10.04 in)

g (ped) (ped

Features for 2651:

• Radius - width ratio min. 1.0 for tight curves **Weight:** 8,8 kg/m² (1.8 lbs/ft².)

Features for both:

- Heavy-duty extra strong design
- For left or right L- or U-turns
- Optimum combination of materials
- Optimum product support
- Easy installation and maintenance

Pin Material: PBT white Standard length:

width ≤ 850 mm (33.46 in): 1,524 m (5 ft.) width > 850 mm (33.46 in): on request

PATENTED

3 0 7 6 0 a





Information about this product is only available on our Smart Guide® website.

Standard material

LFG (GREY)

Low friction acetal resin

materials details:



Also available into

LEW (COLOR	WHITE)

Low friction acetal resin



materials details:



WID.	WIDTH W		2451			MAX LOAD CAPACITY				BACK FLEX RADIUS	
mm	in	ITEM DESCRIPTION	WID	TH W	SIDE FLEX	STRA	IGHT	CUI	RVE		<u> </u>
			mm	in	R. MIN.	N/m	lbs/ft.	N	lbs	mm	in
-	-	=	-	-	-				790	25	
340	13.39	LFG2451TAB-M0340	340	13.39	545		0000	3500			0.98
425	16.73	LFG2451TAB-M0425	425	16.73	680						
510	20.08	LFG2451TAB-M0510	510	20.08	840	20000					
595	23.43	LFG2451TAB-M0595	595	23.43	980	30000	2000				
680	26.77	LFG2451TAB-M0680	680	26.77	1150						
765	30.12	LFG2451TAB-M0765	765	30.12	1300						
850	33.46	LFG2451TAB-M0850	850	33.46	1450						

Side flex radius values are valid for curves up to 180 degrees. For larger angles or for spirals, please consult our Application Engineering.

WIDTH W			2651		MAX LOAD CAPACITY				BACK FLEX RADIUS		
mm	in	ITEM DESCRIPTION	WID	TH W	SIDE FLEX	STRA	IGHT	CUI	RVE		
			mm	in	R. MIN.	N/m	lbs/ft.	N	lbs	mm	in
-	-	-	-	-	-						
425	16.73	LFG2651TAB-M0425	425	16.73	425		2000	3500	3500 790	25	0.98
510	20.08	LFG2651TAB-M0510	510	20.08	510						
595	23.43	LFG2651TAB-M0595	595	23.43	595	30000					
680	26.77	LFG2651TAB-M0680	680	26.77	680						
765	30.12	LFG2651TAB-M0765	765	30.12	850						
850	33.46	LFG2651TAB-M0850	850	33.46	1050			į l			

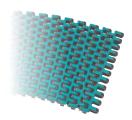




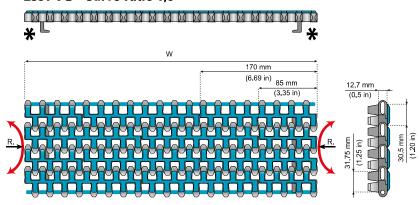
31,75 mm (1 1/4") MODULAR BELTS 2351 - RUBBER INSERT





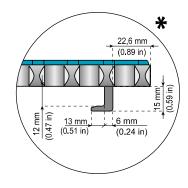


2351 VG - Curve ratio 1,6



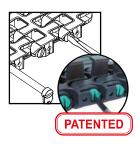


- Radius width ratio min. 1,6
- For left or right L- or U-turns
- Can also be used in S-curves
- Strong design
- Optimum product support
- Easy installation and maintenance



Rubber material: thermoplastic rubber, waterblue colour, 75 shore A. **Standard length:** 96 pitches (3,048 m - 10 ft.)





WID.	TH W		MAX LOAD CAPACITY			BACK FLEX RADIUS		WEIGHT			
		SIDE FLEX R. MIN.	STRA	IGHT	CUI	RVE					
mm in		N/m	lbs/ft.	N	lbs	mm	in	Kg/m²	lbs/ft².		
255	10.04	408									
340	13.39	545									
425	16.73	680									
510	20.08	840	22000	22000 150	1500	0500	2500 560	25	0.98	8,8	1.8
595	23.43	980			1500	2500					
680	26.77	1150									
765	30.12	1300									
850	33.46	1450									

Side flex radius values are valid for curves up to 180 degrees. For larger angles or for spirals, please consult our Application Engineering.

Standard material

LFG (GREY)		
Low friction acetal resin		

materials details:



Also available into

LFW (COLOR WHITE)



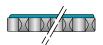
TPR (WATER BLUE) Thermoplastic rubber

materials details:

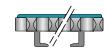




Information about this product is only available on our Smart Guide® website.







With TAB 2 sides

WIDTH W		ITEM DESCRIPTION	ITEM DESCRIPTION	
mm	in	ITEM DESCRIPTION	ITEM DESCRIPTION	
255	10.04	LFG2351-M0255VG	LFG2351TAB-M0255VG	
340	13.39	LFG2351-M0340VG	LFG2351TAB-M0340VG	
425	16.73	LFG2351-M0425VG	LFG2351TAB-M0425VG	
510	20.08	LFG2351-M0510VG	LFG2351TAB-M0510VG	
595	23.43	LFG2351-M0595VG	LFG2351TAB-M0595VG	
680	26.77	LFG2351-M0680VG	LFG2351TAB-M0680VG	
765	30.12	LFG2351-M0765VG	LFG2351TAB-M0765VG	
850	33.46	LFG2351-M0850VG	LFG2351TAB-M0850VG	





DRIVE AND RETURN SPROCKETS, IDLER WHEELS

Smart

www.SystemPlastSmartGuide.com

FOR BELTS 2351 (VG)-2451-2551-2651

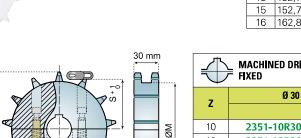
Set collars, go to:



Features:

- Very high strength.
- Split versions: quick and easy replacement.
- Foolproof design.
- Completely closed structure, easy to clean and to disinfect.
- Excellent wear and chemical resistance.
- Recyclable.





Z	Ø P mm	Ø OD mm	Ø M mm	S mm	DIAMETER OF THE SUPPORTING WHEEL mm
10	102,8	107,7	84	45	85,0
13	132,7	138,8	114	60	116,1
15	152,7	159,4	135	70	136,7
16	162,8	169,6	145	75	146,9

MACHINED DRIVE SPROCKETS - SPLIT FIXED			
Z	Ø 30	Ø 40	
	ITEM DES	ESCRIPTION	
10	2351-10R30M-DMS	2351-10R40M-DMS	
13	2351-13R30M-DMS	2351-13R40M-DMS	
15	2351-15R30M-DMS 2351-15R40		
16	2351-16R30M-DMS	2351-16R40M-DMS	

Material: polyamide (natural white), screws in stainless steel, bushings in brass, DIN 6885 key seat.

Ø P Ø OD

See belts 2351-2551:

3 0 7 4 5 a

See belts 2651-2451:

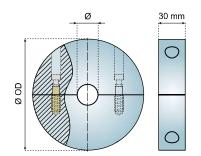
3 0 7 5 0 a

See belts 2351VG:

3 0 7 5 5 a

MACHINED RETURN SPROCKETS - SPLIT			
Z	Ø 30	Ø 40	
	ITEM DESCRIPTION		
10	2351-10R30M-RMS	2351-10R40M-RMS	
13	2351-13R30M-RMS 2351-13R40M-RMS 2351-15R30M-RMS 2351-15R40M-RMS		
15			
16	2351-16R30M-RMS	2351-16R40M-RMS	

Material: polyamide (natural white), screws in stainless steel, bushings in brass.



	MACHINED IDLER WHEE SUPPORTING WHEELS	lτ		
Z PART NUMBER		Ø	NOTES	
10	2351-10R18M-WMS	18	Max. allowed bore: Ø30	
13	13 2351-13R23M-WMS			
15	2351-15R23M-WMS	23 Max. allov	Max. allowed bore: Ø50	
16	2351-16R23M-WMS	2351-16R23M-WMS		

16	2351-16R23M-WMS			
Materia	I: polyamide (natural whit	e), screv	vs in stainless steel,	
bushings	s in brass			

Ø OD mm

85,0

116,1

136,7

146,9

10

13

15

16





PRODUCT HANDLING

GENERAL CALCULATION



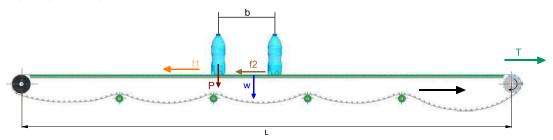
CHAIN/BELT PULL CALCULATION FOR GENERAL PURPOSE CONVEYOR

The pull required to convey the products can be calculated for several conveyor types as follows:



A- STRAIGHT FLAT CONVEYOR WITHOUT ACCUMULATION

The total pull at drive end is the sum of return part pull and carry part pull. See below scheme for involved forces representations at conveyor carry and return parts.



- L: Length of the conveyor shaft to shaft (m or ft);
- b: Product pitch, not required when product weight per unit length is noted;
- P: Product weight per unit length (N/m or lbf/ft):
- W: belt/chain weight per unit length (N/m or lbf/ft);
- g: gravity (m/s² or ft/s²);

The mass of chain per unit length is provided for each chain on SmartGuide® catalog;

For belt the mass per unit square is available on SmartGuide® catalog; the mass per unit length is done by multiplying the mass per unit square by the width of the belt in [m] or [ft].

CARRY PART:

The carry part pull Tc is the sum of f_1 and f_2 :

(1) $Tc = (f_1 + f_2) [N \text{ or } lbf]$

 \mathbf{f}_1 : friction force between belt/chain and wear strips due to the product weight

 \mathbf{f}_2 : friction force between belt/chain and wear strips due to belt/chain weight

(2) $f_1 = \mu_{cp} *P * L$

 μ_{cn} : coefficient of dynamic friction between product and belt/chain due to product

(3) $f_2 = \mu_{cw} * W * L$

 μ_{cw} : coefficient of dynamic friction between belt/chain and wear strip due to belt/chain

Equation (1) becomes:

(4) $T_c = (M * \mu_{cp} + M_c * \mu_{cw}) * g * L [N \text{ or lbf}]$

 M_c : Mass of chain/belt per unit length [kg/m or lb/ft]

M: Mass of the product per unit length [kg/m or lb/ft]

NOTE: In normal condition we assume the product for not slide over the chain/belt so $\mu_{cp} = \mu_{cw}$. For other conditions where product slides over the chain/belt as accumulation $\mu_{cp} \neq \mu_{cw}$.

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continue >

7 3 5 1 9 a Product national maximum acceleration

Pressure of accumulation products

7 3 5 2 4 a 90° product transfer with belts

7 3 5 2 6 a Thermal expansion

7 3 5 2 8 a 2500RR calculations

7 3 5 4 7 a Bearing and shaft

7 8 a Coefficient of friction of standard materials



7 3 5 3 3 a 2253RTC calculations





PRODUCT HANDLING

GENERAL CALCULATION



RETURN PART

In the return part there is no product carried, thus the mass of the product per unit lenght M=0.

The return part pull T_{R} is then:

(5) $T_R=g*L*Mc*\mu_{cr}$ [N or lbf]

 μ_{cr} : coefficient of dynamic friction between belt/chain and guide in the return part.

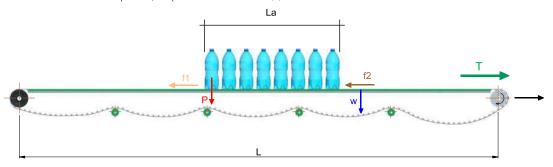
The total pull T of the flat conveyor (excluded the pull due to chordal effect) is done by:

(6)
$$T=T_C+T_R=g*L*(M*\mu_{cp}+M_c\;(\mu_{cw}+\mu_{cr}))$$
 [N or lbf]

The coefficient of friction can be monitored with the iCOF® device. For more details refer to 5 7 0 8 0 a

B- STRAIGHT FLAT CONVEYOR WITH ACCUMULATION:

In case of accumulation of the product, the pull will be as in formula (6) added the contribution of accumulation section.



Considering:

La: length of the accumulation section [m or ft];

 M_a : mass per unit length of accumulated products [kg/m or lb/ft];

 μ_{ep} friction coefficient between belt/chain and accumulated products;

The COFs between chain/belt and wearstrips and between chain/belt and products are available at page 7 3 5 7 8 a The Total pull will be:

(6a)
$$T=T_C+T_R+T_a=[g*(L-L_a)*(M*\mu_{cp}+M_c*\mu_{cw}]+[g*L*M_c*\mu_{CR}]+[M_a*\mu_{cp}*g*L_a]$$











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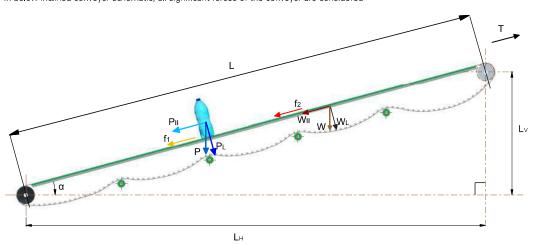


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B- STRAIGHT INCLINED CONVEYOR

As the only difference between an inclined and a flat conveyor is the angle of inclination α , we will skip most calculation steps to get in the final formula.

In below inclined conveyor schematic, all significant forces of the conveyor are considered



 α = incline angle (°);

CARRY PART

The carry part pull T_c is the sum of all horizontal forces;

(7)
$$T_c = (f_1 + P_{II} + f_2 + W_{II})$$

where

(8)
$$f_1 = \mu_{cp} * P_L * L = \mu_{cp} * P * L * \cos \alpha$$

(9)
$$f_2 = \mu_{cw} * W_L * L = \mu_{cw} * W * L * \cos \alpha$$

(10)
$$P_{II} = P *L *sin \alpha = M *g *L *sin \alpha$$

(11)
$$W_{II} = W *L *sin \alpha = Mc *g *L *sin \alpha$$

<u>RETURN PART</u>

In the return part there is no product carried, thus M=0.

(12)
$$T_r = M_c *g *L *(\mu_{cr} * \cos \alpha + \sin \alpha)$$

The total pull T of the inclined/declined conveyor (excluded the pull due to chordal effect) is done by:

(13)
$$T=T_C+T_R=g*L*[M*(\mu_{cp}*\cos\alpha+\sin\alpha)+M_c*(\mu_{cw}*\cos\alpha+\sin\alpha)]+g*L*M_c*(\mu_{cr}*\cos\alpha+\sin\alpha)$$

C- NET SHAFT POWER NP

(14) NP=T*v

V is the conveyor speed [m/s or ft/s]

T the pull [kN or lbf]

NP [kW or lbf*ft/s]

Above calculation is the basic, for detailed calculation use our calculation program SPEC® (http://spec.systemplast.com)

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PRODUCT HANDLING



GENERAL CALCULATION



MAXIMUM ACCELERATION:

The maximum acceleration force F_{max} on a product to be able to convey the product with the chain depends on the friction between product and chain.

Maximum acceleration ${f a}_{max}$ can be calculated with:

$$a_{max} \ = \frac{F_{max}}{M} = \frac{W * \mu}{M} = \frac{M * g * \mu}{M} = g * \mu$$

W = weight of product in [N] or $[Lb_f]$

M = weight of product in [kg] or $[lb_m]$

 μ = coefficient of friction between chain/belt and product

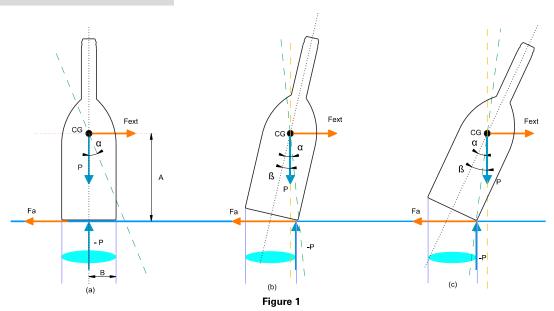
g = gravitational acceleration = 9.81 m/s² (32.2 ft/s²)

MAXIMUM FORCE ON PRODUCTS TO AVOID TIPPAGE:

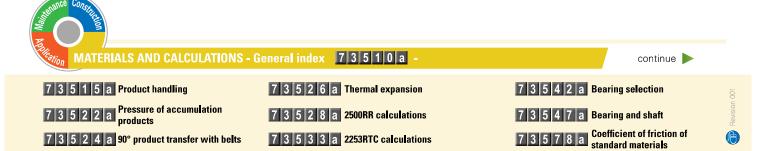
The maximum acceleration a_{cs} without products falling over is dependent on the shape (position of the center of gravity), the weight, and the product material. This is for instance, it is also important when the product is being conveyed onto a dead plate.

When a bottle is moving on conveyors, there are external forces acting on it:

$$F^{ext} = M * a_{CG} = \mu * M * g$$



The force $\mathbf{F}^{\mathbf{ext}}$ is the force due to acceleration or deceleration of the product or due to a different cause like other bottles or a side guide.



Contacts: EUROPE and ROW: SystemPlastTechSupport@regalbeloit.com; USA: systemplast.engineering@regalbeloit.com; Engineering calculator:





PRODUCT HANDLING

GENERAL CALCULATION



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BOTTLE EQUILIBRIUM:

The equilibrium condition of a bottle is that the vector of the vertical component of resultant forces passing by its gravity center falls inside support base (blue circle) i.e. $\alpha > \beta$. See figure 1.

Angle α is determined by the diameter of the footprint of the product (B= ½ * bottle diameter) and the height of the centre point of gravity

Angle β is determined by the horizontal force on the bottle (F^{ext}) relative to the weight of the bottle (P).

Bottle falls when $\beta > \alpha$. That means in:

$$\frac{F^{\text{ext}}}{P} > \frac{B}{A} \; ; \; \frac{m * g * \mu}{m * g} > \frac{B}{A} \; \Rightarrow \; \; \mu > \frac{B}{A}$$

In conclusion:

when
$$\,\mu>\frac{B}{A}$$
 containers fall; and when $\,\mu<\frac{B}{A}\,$ containers remain standing

CENTRIFUGAL FORCES:

When a product is being conveyed through a curve there's a centrifugal force working on the product. This force on the product is compensated by the friction between chain and product and by a side guide.

The centrifugal force is calculated with:

$$F_c = \frac{M * v^2}{r}$$

M = weight of the product

V = speed

r = center radius of the curve

Friction force between chain and product is calculated with:

$$F_m = M *g *\mu$$

g = gravitational acceleration

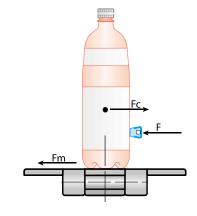
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 μ = coefficient of friction between belt/chain and product

The minimum force ${f F}$ that needs to be generated by the side guide is:

$$F = F_c - F_m = M * \left[\frac{v^2}{r} - g * \mu \right]$$





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PRESSURE OF ACCUMULATION PRODUCTS

GENERAL CALCULATION



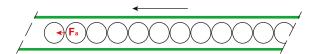
PRESSURE OF ACCUMULATING PRODUCTS OR BACKLINE PRESSURE:



When a product is standing still (e.g. against a stopper or guide), the chain running underneath the product creates a force on the product equal to the weight of the product multiplied by the coefficient of friction between chain and product.

Each following product is pushing with the same force against the next product, so the resulting force is proportional to the total weight of products upstream. This force is the same of the accumulation contribute on formula 6A.

$$\begin{aligned} F_a &= W_a * L_a * \mu \\ F_a &= M_a * g * \mu_{cp} * L_a \end{aligned}$$



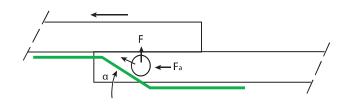
 F_a = accumulation force

 $\mathbf{W}_{\mathbf{a}}$ = weight of the accumulating products in Kg/m (or lb/ft).

 L_a = length of accumulation in m (or ft)

 μ = coefficient of friction between chain and product.

SIDE TRANSFER ACTION:



Pushing the product sideward creates a force F on the product against the side guide.

$$F = F_a * sin(\alpha) = W_a * L_a * \mu * sin(\alpha)$$

(see explanation of symbols above)

Nowadays, can and bottle wall thickness is becoming thinner and thinner. At the same time, more installations are running with less or no lubrication, thus increasing the coefficient of friction (unless they utilize our NGE / Nolu®-S parts solution). That's why it's important to take also the forces on the products into consideration.

In the above-mentioned formula, the angle α plays an important role in a smooth transfer and reduced forces on the products. This angle should be kept a small as possible. One should also consider the nesting pattern of cans and bottles during a side transfer to optimize guide rail geometry.

CALCULATION OF SIDE GUIDE DISTANCE:

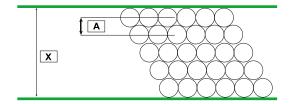
SIDE GUIDE DISTANCE X = D + (N-1) * A

Where:

D = Product diameter

N = Number of product rows

 $A = \text{product-product center distance} = \sqrt{0.75 * D^2}$



— SIDE GUIDE



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90° PRODUCT TRANSFER WITH BELTS

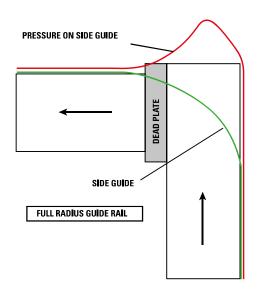


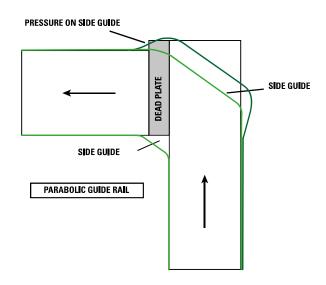


GENERAL CALCULATION

90° PRODUCT TRANSFER:

When transferring products from one conveyor to another at a 90° angle, it's common practice to use full radius guide rails with dead plates which span the space between the infeed and outfeed conveyors. Products moving along the full radius guide rail exert high pressure on the rail and on each other, easily resulting in damage.





The parabolic guide rail distributes the pressure better along the outer guide rail. This results in significantly less potential for damage on products or deflection of the guide rail. However, on the inside corner of the curve, extra room must be created for an extended dead plate "dead area".

The use of the System Plast® active transfer wing system will eliminate the dead plate and further reduce the pressure on products and side guides.

Belt with transfer wing. Go to:



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THERMAL EXPANSION

GENERAL CALCULATION







THERMAL EXPANSION

Due to temperature variation, the belt width can change according to the delta temperature. The belt width variation is function of the delta temperature and the belt material itself. See below formula for thermal expansion calculation.

CALCULATION:

Expected expansion ΔW [mm or in];

$$\Delta \mathbf{W} = \mathbf{W} * \Delta \mathbf{T} * \mathbf{f}$$

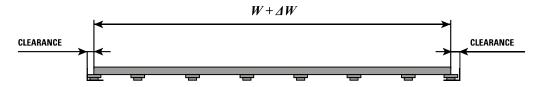
W = belt width [m] or [in]

 ΔT = temperature difference (Operating temperature - Ambient temperature at manufacturing site) [°C] or [°F]

 \mathbf{f} = thermal expansion factor (material property) or Linear Thermal Expansion Coefficient [mm/m/°C] or[in/in/°F]

MATERIAL	THERMAL EXPANSION FACTOR F		
MATERIAL	[mm/m/°C]	[in/in/°F]	
LF (Acetal)	0.12	6.66E-5	
NG [®] evo conveyor components made from engineered plastic resin (PBT)	0.13	7.22E-5	
PP (Polypropylene)	0.15	8.33E-5	
HT (PA)	0.08	4.44E-5	
XT (PPS)	0.26	1.44E-4	

To be able to absorb the thermal expansion the belt needs some clearance on the side. Depending on the expected temperature difference the structure should be wide enough to make sure the belt will not get stuck at high temperatures or, in case of low temperatures, be small enough to still support the belt at low temperatures.



The total clearance per side to be considered is (indicative values):

BELT NOMINAL WIDTH		TOTAL CLE	ARANCE (*)
W < 500 mm W < 20"		4 mm	0.16"
500 < W < 1500 [mm]	500 < W < 1500 [mm] 20" < W < 60"		0.24 ''
1500 < W < 3000 [mm]	60" < W < 120"	8 mm	0.32"
W > 3000 mm W > 120"		10 mm	0.39"

(*) For PP belts, please consider the extra dimension due to material expansion during molding process in the clearance calculations.

NOMINAL WIDTH TOLERANCE

The standard tolerance on the nominal width of belts/chains made in Acetal (LF) or Polybutylene terephthalate (NGE) is +0 / - 0,5 %. The standard tolerance on the nominal width of belts/chains made in Polypropylene (PP, PP-FR-EC, CR) is +1 % / +2 %.

The standard tolerance on the nominal width of belts/chains made in Acetal (POM-AS) is 0 / +0.6 %.



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RAISED RIB BELT 2500



CONSIDERATION OF THERMAL INFLUENCES

continue

- 1. Calculate the thermal width expansion " ΔlW "
- 2. Calculate the thermal belt elongation "ΔIE"
- 3. Determine the catenary height "H"

NOTE: THESE CONSIDERATIONS ARE NOT INCLUDING THE EXPANSION INFLUENCES DUE TO BELT TENSION!

2500RR Construction details. Go to: 7 2 3 1 0 a

Thermal expansion calculation:

Calculation: Expected expansion [mm] $\Delta l = W \cdot \Delta T \cdot f$

= thermal expansion factor (of PPG: ~ 0,15 mm/m/°C or 8.33E-5 in/in/°F)

= belt width [mm]/1000 or belt length per zone [mm]/1000

 ΔT = temperature difference = ([°C] - 21°C)

1. Calculate the thermal width expansion " $\Delta lW^{\prime\prime}$

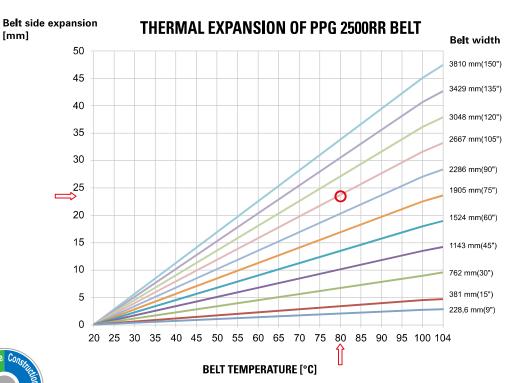
Example for a belt 105" (2667 mm) width, at 80°C

 $\Delta \mathbf{l} = \mathbf{W} \bullet \Delta \mathbf{T} \bullet \mathbf{f}$

 $\Delta l = 2667 \text{ mm}/1000 \text{ x } (80^{\circ} - 21^{\circ})\text{C x } 0.15 \text{ mm/m/}^{\circ}\text{C}$

 $\Delta IW = 23 \text{ mm}$

Thermal width expansion can also be read out of below diagram.



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RAISED RIB BELT 2500





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2. Calculate the thermal belt elongation "ΔΙΕ" for e.g. different temperature zones in a warmer tunnel

Example for a warmer-tunnel, with a 17,5 m shaft-distance, standard conveyor design with head drive, and three temperature

zones. Zone 1:5 m long, 40°C

Zone 2: 7,5 m long, 80°C Zone 3:5 m long, 50°C

Note: Don't forget to count also the amount of belt of the return part, where the belt is also exposed to the several temperatures! This leads to double the elongation of the belt in each zone!

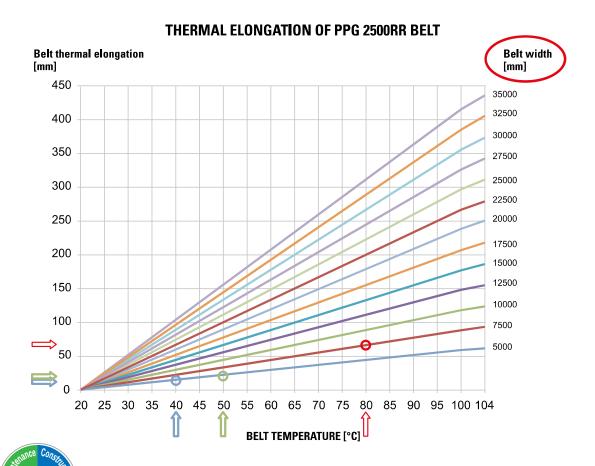
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 $\Delta IE = \Delta IZone1 + \Delta IZone2 + \Delta IZone3$ Δ|Zone1 = WZone1 • ΔTZone1 • f = 5 m x (40 - 21)°C x 0,15 mm/m/°C = 14,3 mm x 2 = 28,6 mm $\Delta | Zone2 = WZone2 \cdot \Delta TZone2 \cdot f = 7.5 \text{ m x } (80 - 21)^{\circ} \text{ C x } 0.15 \text{ mm/m/}^{\circ} \text{ C} = 66.4 \text{ mm x } 2 = 132.8 \text{ mm}$ $\Delta IZone3 = WZone3 \cdot \Delta TZone3 \cdot f = 5 \text{ m x } (50 - 21)^{\circ}C \times 0.15 \text{ mm/m/}^{\circ}C = 21.8 \text{ mm x } 2 = 43.6 \text{ mm}$ $\Delta IE = 28,6 \text{ mm} + 132,8 \text{ mm} + 43,6 \text{ mm}$ Δ**I**E = 205 mm

Thermal belt elongation can also be read out of below diagram.





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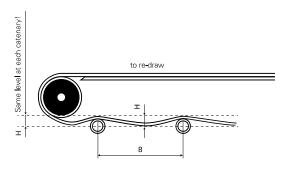


RAISED RIB BELT 2500

3. Determine the catenary height "H"

Example based on a belt elongation of ΔIE = 205 mm, from thermal belt elongation example go to page code: 7 3 5 2 9 a determined by calculation or by using the diagram for thermal elongation go to page code: 73529a)





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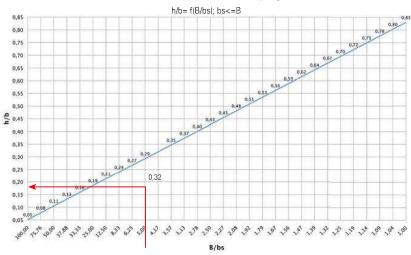
 $= \mathbf{B} \mathbf{x} \mathbf{h}/\mathbf{b}$ H

В = expected catenary length / distance between return roller (read out of drawing from the conveyor)

= factor determined from the following diagram

= belt stretch (thermal elongation ΔIE from thermal belt elongation example go to page code: 7 3 5 2 9 a Determined by calculation or by using the diagram for thermal elongation go to page code: 7 3 5 2 9 a

> catenary height as a function of belt stretch for all catenary lengths and as long as belt stretch is smaller than the catenary length



Example:

 $\mathbf{bs} = \Delta \mathbf{IE} = 205 \text{ mm}$

B $= 1000 \, \text{mm}$

 $B/bs = 1000 \text{ mm} / 205 \text{ mm} = 4,88 \approx 5$

 $\mathbf{h/b} = 0.29$ (read out of diagram)

 $H = B \times h/b = 1000 \text{ mm} \times 0.29 = 290 \text{ mm}$

Note: Consider a minimum tolerance of +/-30 mm for the calculated catenary height

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2500RR LOAD CAPACITY DEPENDING ON APPLICATION TEMPERATURE **RAISED RIB BELT 2500**







Construction details.

2500RR

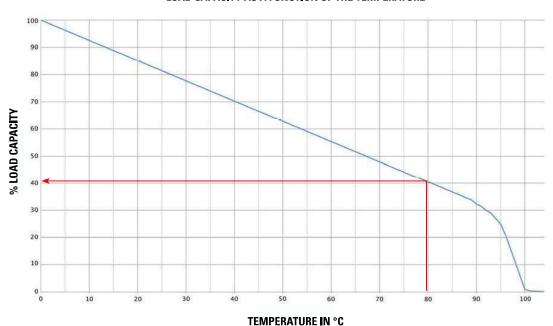
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LOAD CAPACITY / TEMPERATURE GRAPH

NOTE: Mechanical properties of plastic materials degrade with the increasing temperature.

The following graph indicates the remaining load capacity (%) as a function of the temperature.

LOAD CAPACITY AS A FUNCTION OF THE TEMPERATURE



EXAMPLE:

Identify the hottest section. (In the example before: section 2, 80° C)

The allowable max. load capacity/working load of the belt in that particular section is:

Factor 0,41 (read 41% out of diagram) \times 30000 N/m = 12300 N/m.

It can happen that the middle section of a machine is more critical with respect to belt load than the drive section, due to the fact that the temperature at the drive is lower and therefore the allowable belt tension is higher.

This needs to be checked carefully.



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2253 ROLLER TOP BELT

1- LATERAL MOVEMENT

Auxiliary drive units underneath the belt, running at right angles to the direction of belt travel, enable lateral transfers (left- or right side sorting) of conveyed goods. It can also be used to position a product, for example in front of a palletizer. Since no push-off devices, etc., are necessary (more safe), this approach works to prevent jamming, and allows for compact, space-saving layouts and no safety guards are required.



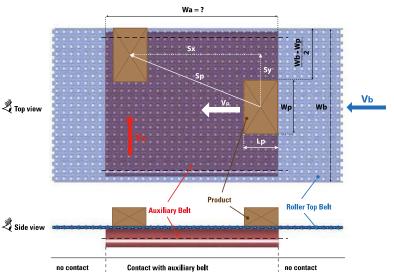
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EXAMPLE HOW TO MAKE A LATERAL MOVEMENT ON THE FLY (*)



GIVEN DATA:

Width of pack ($\mathbf{W}_{\mathbf{p}}$) = 240 mm;

Length of pack $(L_p) = 360 \text{ mm}$

Capacity (P_r) = 30 p/min.

 S_{out} = 150 mm

As a rule of thumbs, for a side shift with only one auxiliary belt, the minimum width of the auxiliary belt (1) should be at least 1,5 times the transversal dimension of the pack Lp.

Bigger widths help in leaving more time to the movement thus reducing the speed of the auxiliary belt and consequently giving a better positioning of the pack (inertia results lower).

Minimum Width of the auxiliary belt (1) (\mathbf{W}_{a}) will be calculated as follow:

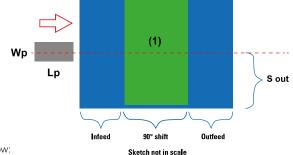
 $W_a = 1.5 \times 360 \text{ mm} = 540 \text{ mm} \approx 600 \text{ mm}.$

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At the same time minimum pitch in feeding, centre to centre pack, needs to be equal or bigger than the auxiliary belt width to avoid packs to be rotated before being shifted. The minimum pitch will be equal to 600 mm.

(*) For stop and go motion the minimum width of the auxiliary belt W_a is equal to the product lenght L_p ($W_{a=}L_p$).



1 m = 3.28 ft

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2253 ROLLER TOP BELT



This pitch must be realized, by means of appropriate feeding/metering conveyors (for instance equipped with VG belts) before the packs

Bigger pitches have the effect to increase the speed of the Roller Top system but not the overall dimension of the application.

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$$V_p = \frac{Pitch}{1000} * P_r = 0.6 * 30 = 18 \text{ m/min}$$

ullet The speed of 2253RTC belt ($f V_{rt}$) can be calculated considering the 2253 Roller Top belt itself running fully on spheres:

$$V_{rt} = \frac{1}{2} * V_p = 0.5 * 18 = 9 \text{ m/min}$$

- Time for the pack to climb on the area (1) (\boldsymbol{t}_{c})

$$t_{c} = \frac{lenght\ of\ pack\ L_{\textrm{\tiny p}}\ [mm]}{pack\ speed\ V_{\textrm{\tiny p}}\ [\frac{m}{min}\]} = \frac{360\ *60}{18\ *1000}\ = 1.2\ s$$

climb on the roller top belt or on the infeed static of the roller top belt.

• Time to cross the auxiliary belt area (1) (t_a)

$$t_{\rm a} = \frac{overall\ lenght\ shift\ area\ [mm]}{pack\ speed\ V_{\rm p}\,[\frac{m}{min}\,]} = \frac{600\ *60}{18\ *1000} = 2\ s$$

Once the pack is fully on the area (1) the shift can start.



If the shift is earlier activated, the pack will be turned/misaligned instead of being shifted.

Note: In case the auxiliary belt is running continuously, the dimensions of the auxiliary bet area as well as the overall footprint can be reduced. In such case a more defined and precise controls of the belts timings are required.

• The time available for the side shift (ts) is given by the difference of the time t_a and t_c

$$t_s = t_a - t_c = 2 - 1.2 = 0.8 \text{ s}$$

The pack is supposed to come on the Roller Top area about in the middle of it.

Consequently, the total travel distance to lead the outer front of the pack completely on the bay area is the sum of S_{out} + half of product width $\mathbf{W}_{\mathbf{p}}$.

ullet Considering this, the auxiliary belt speed V_a results:

$$V_a = \frac{S_{out} + \frac{W_p}{2}[mm]}{t_s[s]} = \frac{150 + 120}{0.8 * 1000} = 0.34 \text{ m/s} = 21 \text{ m/min}$$

The infeed is needed to have a stable pack before the shift (head to tail passage can affect the stability of the pack). It should be at least as big as the pack length.

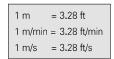
Infeed = L_p = 360 mm \approx 400 mm;

The outfeed is recommendable but not mandatory. The outfeed dimensions should be in the same dimension range of the infeed.

Outfeed = Infeed = 400 mm

The overall footprint or length of the roller top and auxiliary conveyor system will be the sum of infeed, outfeed and shift area length.

Overall footprint = (400 + 600 + 400) mm = 1400 mm.



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2253 ROLLER TOP BELT

2- ROTATION WITH PRODUCT STOPPED: TURNING DISC UNDERNEATH THE 2253RTC BELT

The main advantage of this configuration is the easier handling of the packs in terms of settings of drives required.

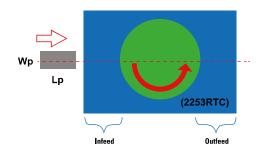
On the other hand it's need of a fairly exact displacement of the pack in the centre of the table area for a good rotation.



EXAMPLE OF HOW TO DIMENSION AN APPLICATION OF ROTATION WITH PRODUCT STOPPED:

GIVEN DATA:

Width of pack (W_p) = 240 mm; Length of pack (L_p) = 360 mm Capacity (P_r) = 30 p/min. Rotation angle (α) = 90°



2253RTC Construction details. Go to:



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For a proper rotation, the diameter of the disk (D_{TD}) should be in the same range of the dimension of diagonal of the pack. Slightly bigger disks help in reducing the need of a precise placing in the centre of it. In case of big diagonal dimension, if possible, reduce the dimension of the disk to not expose ball placed on the outer diameter to too quick accelerations. Movement would result longer since less balls in contact.

• Disk diameter can be calculated as follow:

$$D_{TD} = \sqrt{W_p^2 + L_p^2} = \left(\sqrt{240^2 + 360^2}\right) mm = 432 mm \approx 450 mm$$

At the same time minimum pitch in feeding, centre to centre pack, needs to be equal or bigger than the turning disk diameter. This to avoid the pack to be rotated when the previous pack is still on the turning disk area. The minimum pitch will be equal to 450 mm.

This pitch must be realized, by means of appropriate feeding/metering conveyors (for instance equipped with VG belts) before the products climb on the 2253RTC or on the infeed static of the roller top belt.



Bigger pitches have the effect to increase the speed of the Roller Top system but not the overall dimension of the application.

- The product speed $V_{\mbox{\tiny p}}$ will be as follow:

$$V_p = \frac{\text{Pitch}}{1000} * P_r = 0.45 * 30 = 14 \text{ m/min}$$

• The speed of 2253RTC belt (VRT) can be calculated considering the 2253 Roller Top belt itself running fully on spheres:

$$V_{rt} = \frac{1}{2} * V_p = 0.5 * 14 = 7 \text{ m/min}$$

1 m = 3.28 ft 1 m/min = 3.28 ft/min 1 m/s = 3.28 ft/s



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2253 ROLLER TOP BELT

The time available for the rotation (tr) will be as follow:

$$t_r = \frac{\text{Pitch}}{1000 * V_p} = \frac{450}{1000} * \frac{60}{14} = 1.93 \text{ s}$$

The angular speed ω can be calculated through below formula:

$$\alpha = \omega * t_r$$

So
$$\omega = \frac{\alpha}{t} = \frac{\frac{\pi}{2}}{1.93} = 0.81 \text{ rad/s} = 49 \text{ rad/min}$$



Note: in this configuration, the conveyor must be stopped while the pack is rotating, and the pack should be as close as possible to the centre of the disk.

The infeed is recommended but not mandatory to have a stable product on conveyor before the rotation.

The infeed length should be at least as big as the pack: 360 mm ≈ 400 mm

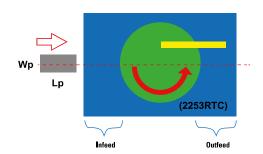
The outfeed is recommended too but not mandatory.

The outfeed length should be in the same dimension range of the dimension of the pack once rotated (i.e Wp): 240 mm ≈ 250 mm

The overall footprint or length of the roller top and turning disk system will be the sum of infeed, outfeed and rotation area diameter.

Overall footprint = (400 + 450 + 250) mm = 1100 mm.

Alternately, by placing a roller guide (in yellow) as shown in the sketch below is possible to keep the disk always in motion avoiding the stop of the conveyor during rotation.











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2253 ROLLER TOP BELT

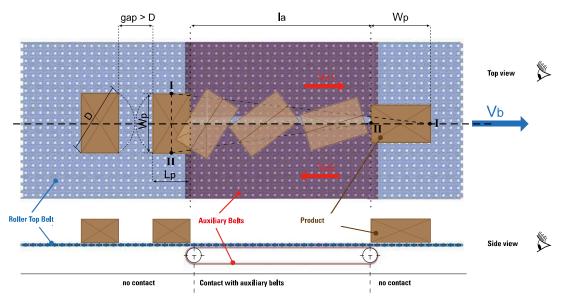
3- ROTATION WITH PRODUCT MOVING: AUXILIARY BELTS UNDERNEATH THE 2253RTC BELT



With two parallel auxiliary belts running underneath the 2253RTC belts at different directions, the packs positioned in between the auxiliary belts will rotate while moving on the 2253RTC belt.



EXAMPLE OF HOW TO DIMENSION AN APPLICATION OF ROTATION WITH PACK MOVING:



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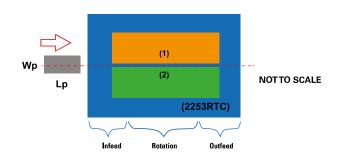
7 2 6 2 5 a

Width of pack (\mathbf{W}_p) = 240 mm;

Length of pack $(L_p) = 360 \text{ mm}$

Capacity (P_r) = 30 p/min.

Rotation angle (α) = 90°



The minimum product pitch, centre to centre pack, needs to be equal or bigger than the product diagonal (D) in order to avoid product touching each other during the rotation.

• The minimum pitch will be equal to

Pitch = D =
$$\sqrt{W_p^2 + L_p^2}$$
 = $(\sqrt{240^2 + 360^2})$ mm = 432 mm ≈ 450 mm

This dimension is also corresponding to the minimum product pitch in feeding, centre to centre pack. This pitch must be realized, by means of appropriate feeding/metering conveyors (for instance equipped with VG belts) before the packs climb on the 2253RTC or on the infeed static of the roller top belt.



1 m = 3.28 ft

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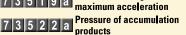
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2253RTC CALCULATIONS

2253 ROLLER TOP BELT



Bigger pitches have the effect to increase the speed of the RollerTop system but not the overall dimension of the application.



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• The pack speed Vp will be as follow:

$$V_p = \frac{Pitch}{1000} * P_r = 0.45 * 30 = 14 \text{ m/min}$$

• The speed of 2253RTC belt (V_{RT}) can be calculated considering the 2253 Roller Top belt itself running fully on spheres:

$$V_{RT} = \frac{1}{2} * V_p = 0.5 * 14 = 7 \text{ m/min}$$

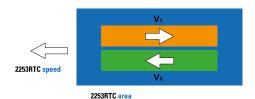
It is recommended to keep the length of auxiliary belts (1) e (2) at least equal 1,5 times the diagonal (D) of the product. So, the minimum length of auxiliary belts (1) and (2) will be:

$$L_a = 1.5 * D = 1.5 * 450 \text{ mm} = 675 \text{ mm} \approx 700 \text{ mm}$$

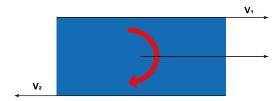
The time to cross (ta) the auxiliary belt area (1) and (2) will be as follow:

$$t_{\rm a} = \frac{\text{overall lenght shift area } L_{\rm a}[mm]}{\text{pack speed } V_{\rm p} \left[\frac{m}{\text{min}}\right]} = \frac{700}{1000} * \frac{60}{18} = 2.33 \text{ s}$$

Considering V_1 and V_2 the speeds given to the spheres by the auxiliary belts (1) and (2) underneath.



The speed of the gravity centre of the product is 7 m/min.



ullet Considering the angular speed ω of the pack the following relations are valid:

$$\alpha = \omega * t_a$$

$$So \omega = \frac{\alpha}{t_a} = \frac{\frac{\pi}{2}}{2} = 0.79 \text{ rad/s} = 47 \text{ rad/min}$$



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maximum acceleration

Pressure of accumulation

1 m = 3.28 ft 1 m/min = 3.28 ft/min 1 m/s = 3.28 ft/s

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2253 ROLLER TOP BELT

The required difference speed to rotate the product will be:

$$(V_1 - V_2) = 0.79 * 225 = 0.17 \text{ m/s} = 11 \text{ m/min}$$

continue >

2253RTC Construction details.

2253RTC

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Another way to calculate the difference in speed of balls can be used. See following relation:

Considering V_{a1} , the speed given to the auxiliary belt (1) and opposite in respect to V_1 (actual speed of balls under auxiliary belt (1));



Similarly, but opposite in running direction is V_{a2} :



Thus:

$$(V_1 - V_2) = 2 * V_{RT} + V_{a1} - (2 * V_{RT} - V_{a2}) = V_{a1} + V_{a2} = 11 \text{ [m/min]}$$

From the above relation is therefore possible to determine the real speeds Va1 and Va2 of the auxiliary belts by combining two speeds at choice V_1 and V_2 .

For instance, considering fixed the difference in speed (V_1-V_2) :

$$(V_1 - V_2) = 11 \text{ m/min}$$
;

$$V_1 = 20 \text{ m/min}$$
;

$$V_2 = V_1 + 11 \text{ m/min} = (20 + 11) \text{ m/min} = 31 \text{ m/min}.$$

The auxiliary belt (1) will run in the same sense of the main belt with a speed Va1.

$$V_{a1} = V_1 - 2 * V_{RT} = (20 - 7 * 2) \text{ m/min} = 6 \text{ m/min}$$

While the auxiliary belt (2) will run in the opposite sense of the main belt with a speed V_{a2} = 31 - 7*2 = 17 m/min

$$V_{a2} = V_2 - 2 * V_{RT} = (31 - 7 * 2) \text{ m/min} = 17 \text{ m/min}$$

Go to: 7 2 2 9 0 a

Applications. Go to: 7 2 6 2 5 a

= 3.28 ft1 m 1 m/min = 3.28 ft/min 1 m/s = 3.28 ft/s

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2253 ROLLER TOP BELT

In case of use of two speeds opposite to the running direction of the product (Fig. A), the following relation is valid:

$$(V_1 - V_2) = 2 * V_{RT} + V_{a1} - (2 * V_{RT} + V_{a2}) = V_{a1} - V_{a2}$$
 m/min

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This configuration requires higher speeds of the auxiliary belts and has the main advantage of the better control and stable product rotation.

At last, it is also possible to use a fixed plate instead of one of the two auxiliary belts (Fig. B).

This solution makes application cheaper (one less drive and auxiliary belt) but has the disadvantage to lead to an higher speed of the auxiliary belt, thus potential higher wear rate.

The infeed is needed to have a stable pack before the rotation (head to tail passage can affect the stability).

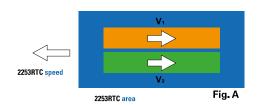
The infeed length should be at least as big as the product length: 360 mm ≈ 400 mm

The outfeed is recommended too but not mandatory.

The outfeed length should be in the same dimension range of the dimension of the pack once rotated (Wp): 240 mm ≈ 250 mm.

The overall footprint or length of the roller top and auxiliary belts system will be the sum of infeed, outfeed and shift area length.

Overall footprint = (400 + 700 + 250) mm = 1350 mm.









Note: The SSL (Short Side Leading) rotation is generally less efficient than LSL (Long Side Leading) since the number of balls in contact at the beginning as well as arm of rotation is smaller.

Rotation could therefore seem not to start immediately and then be sudden.

In case of high speed differentials of (V_1-V_2) is recommendable to increase the auxiliary belts area (the length) to speeds lower and thus realize a smoother rotation.

For a properly working rotation we advise a length/width ratio of max. 2

1 m = 3.28 ft1 m/min = 3.28 ft/min= 3.28 ft/s1 m/s

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BEARING

BEARING SELECTION:

The following general information will serve the purpose of aiding the machine designer or bearing user when applying the bearings covered in System Plast® catalog.

Before beginning the bearing selection for a particular application, it is important to have a good idea of where the bearing will be installed, what its purpose will be, what operating conditions will the bearing be expected to function in, and a desired bearing life. Each bearing type has certain characteristics which make it suitable for a certain application(s). Having comprehensive knowledge of these requirements will aid in bearing selection. In most cases, there are several factors to consider when choosing a bearing type. Therefore, the following information is to be used only as a guide. In the selection process the following factors must be considered:

magnitude and direction of load; misalignment (static or dynamic), noise; vibration and shock loading, environment and bearing type.





LOAD - MAGNITUDE AND DIRECTION

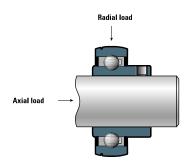
Load magnitude typically dictates size of bearing required but it can also affect the type of bearing. Ball bearings work well in light to moderate loads typical of conveyor applications. Load direction can be radial, axial, or a combination of these two directions. These directions along with load magnitude are deciding factors in selection of bearing type.

System Plast® bearings are mostly used in application where there are only radial load or radial loading plus slight axial load.

Radial loading is the most common type of bearing load and is defined as a load perpendicular to the shaft center line. Most ball bearings are designed to accept primarily radial loads.

Axial loading is defined as loading in the direction through the shaft centerline.

Combination loading consists of both a radial and a thrust load acting simultaneously on the bearing. When combination loads are acting on a bearing it is necessary to determine an equivalent radial load when calculating bearing life.



MISALIGNMENT

Bearing misalignment is a result of angular misalignment between the shaft and housing. This misalignment comes in two different forms, static and dynamic. Static misalignment is the outcome of bearings that are mounted on different planes causing an angular shaft displacement and resulting in the bearing operating under fixed misalignment angle. Mounted ball bearings have a design feature that allows them to accommodate a limited degree of fixed misalignment. Dynamic misalignment is an eccentric shaft rotation caused by shafting imperfections and resulting in the bearing operating under a varying misalignment angle.

Static System Misalignment

Each bearing type is capable of accommodating a certain amount of either static, dynamic, or combination misalignment. When application misalignment exceeds the allowable limit for the particular bearing, increased contact stresses between bearing rolling elements and raceways occurs and bearing life is reduced. Individual product sections contain additional information regarding degrees of misalignment each bearing type is capable of handling.

Dynamic System Misalignment





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BEARING SELECTION

BEARING





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NOISE:

Noise sensitive applications require smooth running of bearings. These are typically low duty environments which makes ball bearings a good choice. Concentric or Skwezloc® locking mechanisms are preferred to keep vibration at a minimum.

VIBRATION AND SHOCK LOADING:

Vibration and shock loading can be present in some applications and can transfer large forces to bearings and accompanying raceways. These loads create large stresses at the interface between the rolling elements and raceways and can cause considerable damage and a reduction of bearing life.

Bearing mounting. Go to: 7 3 0 5 2 a

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ENVIRONMENT:

Environmental factors such as solids contamination exposure to moisture and thermal conditions are important variables in bearing selection. Bearing components (seals, grease, bearing material, etc..) can be modified in order to better suit a specialized application. Availability of special features may be affected by shaft size, bearing type, and housing type therefore this must be considered in the bearing selection process. Individual product sections contain additional information regarding these specialized features and availability.

BEARING TYPE: RADIAL BALL BEARINGS

Radial ball bearings create a fairly small elliptical contact between the ball-path and rolling element thus distributing loads across a small area. Surface contact is minimized and less friction and heat is generated which allows ball bearings a higher speed range. This small contact area also limits ball bearings to accepting only light to moderate loads. Radial ball bearings have a zero-degree free contact angle but can accept light thrust loads (in combination with a radial load) due to the shape geometry of their raceways. Mounted ball bearing units have some degree of external static self-aligning capability (the bearing insert can misalign with respect to the housing). Mounted ball bearings come in a variety of housing styles and features to suit a wide variety of applications. Only "radial" ball-bering type are available from System Plast® product range.



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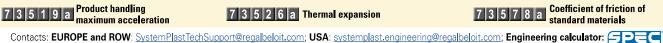
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BEARING SELECTION

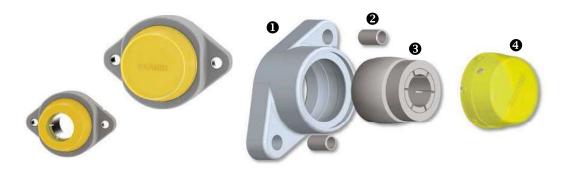
BEARING



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BEARINGS DRAWING EXPLODED VIEW

FREE MAINTENANCE BEARING: TRIATHLON® WASHDOWN BEARING - 2 HOLE VERSIONS - SKWEZLOC® BEARING





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POS.	ITEM DESCRIPTION	QTY
1	Bearing housing machined oval	1
2	Spacer Ø 14mm	2
3	Bearing PN-206T with SKWEZLOC® concentric locking collar	1
4	Safety cap	1

FLANGED HOUSINGS - 2 HOLE VERSIONS - GRUB SCREW LOCK



POS.	ITEM DESCRIPTION	QTY
1	Bearing housing machined oval	1
2	Grease nipple 1/8"	1
3	Bearing with screw	1
4	Spacer Ø 14mm	2
5	Washer	2
6	Heavy ring double lips	1
7	Safety cap	1
8	O-ring	1



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BEARING SELECTION



BEARING

FLANGED HOUSINGS - 4 HOLE VERSIONS - ECCENTRIC LOCK







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POS.	ITEM DESCRIPTION	QTY
1	Bearing Housing	1
2	Grease nipple 1/8"	1
3	Bearing with eccentric ring	1
4	Spacer	4
5	Washer	4
6	Heavy ring double lips	1
7	Safety cap	1
8	O-ring	1



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BEARING AND SHAFT

BEARING

SHAFT SIZE:

THE SHAFT MUST FULFILL THE FOLLOWING CONDITIONS:

- Maximum shaft deflection Fmax under full load (Fw) is 2,5 mm (0.10 in). If the calculated shaft deflection exceeds this max value, select a bigger shaft size or install an intermediate supported bearing on the shaft.
- Torque (Ts) at maximum load must be below critical value; Shaft maximum deflection under concentrated load applied in the center can be calculated with following formula:



Bearing mounting. Go to:

2 BEARINGS

$$f = \frac{5}{384} * \text{Fw} \frac{\text{I}_{b}^{3}}{\text{E} * \text{I}} \text{ [mm]: Fw [N], I}_{b} \text{ [mm], I [mm^{4}], E } \left[\frac{\text{N}}{\text{mm}^{2}} \right]$$

$$f = \frac{5}{384} * \text{Fw} \frac{\text{I}_{b}^{3}}{\text{E} * \text{I}} \text{ [in]: Fw [lb], lb [in], I [in^{4}], E } \left[\frac{\text{lb}}{\text{in}^{2}} \right]$$

3 BEARINGS

$$f = \frac{1}{370} * Fw \frac{I_b^3}{E * I} [mm]; Fw [N], Ib [mm], I [mm^4], E \left[\frac{N}{mm^2} \right]$$

$$f = \frac{1}{370} * Fw \frac{I_b^3}{E * I} [in]; Fw [lb], Ib [in], I [in^4], E \left[\frac{lb}{in^2} \right]$$

For uni-directional head drive $\mathbf{F}_{\mathbf{w}} = \mathbf{T}_{\mathbf{s}}$

For bi-directional center drive $F_w = 2 * T_s$

For uni-directional pusher drives $F_w = 2.2 * T_s$

For modulus of elasticity (E) use: 200.000 N/mm² or 28.500.000 lb/in². Values are for carbon steel (C50) or stainless steel AISI 304/316. Modulus of elasticity for other materials can be referenced via industry standards.

MOMENT OF INERTIA FOR SOLID SHAFTS (I)					
SHAF	T SIZE	INERTIA			
mm	mm in.		in⁴		
SQUARE SHAFTS					
25	-	32552	-		
40	-	213333	-		
60	-	1080000	-		
90	-	5467500	-		

Moment of inertia can be calculated for all shaft diameter and shape. It's a geometrical property of the shaft.

MOMENT OF INERTIA FOR SOLID SHAFTS (I)				
SHAF	T SIZE	INE	RTIA	
mm	in.	mm ⁴	in⁴	
	ROUND	SHAFTS		
20	-	7850	-	
25	-	19150	-	
30	-	39750	-	
35	-	73650	-	
40	-	125600	-	
45	-	201200	-	
50	-	306650	-	
55	-	449000	-	
60	-	635850	-	
80	-	2009600	-	
90	-	- 3220000		
-	1	-	0.049	
-	1.1875	-	0.098	
-	1.25	-	0.120	
-	1.4375	- 0.210		
-	1.5	-	0.248	

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BEARING AND SHAFT



BEARING CALCULATIONS FOR SERIES UC AND UF:

STATIC LOAD CALCULATION:

In case the bearing is loaded without rotating, very slowly rotating or is making a slow oscillating movement, the bearing power is not determined by the fatigue life of the material but by the deformation of the rollers and the groove. This calculation is also valid if at a fraction of the rotation a shock load is present.





Go to:

 $P_0 = X_0 * F_r + Y_0 * F_a = 0.6 * F_r + 0.5 * F_a$

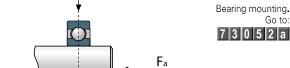
P₀ equivalent static load [N];

X₀ radial static factor;

 \mathbf{F}_{r} radial load [N];

Y₀ axial static factor;

F_a axial load [N]:



 \mathbf{P}_0 is calculated when there is not only a radial component in the load but also an axial component.

When the load on the bearing is strictly radial the equivalent static load $P_0 = F_r$.

The minimum static load coefficient C_0 is calculated while taking into account the static safety factor S_0 .

Then in the bearing tables the right size bearing can be found checking the ${f C}_0$.

$C_0=S_0*P_0$

C₀ static load coefficient in [N]

 S_0 static safety factor;

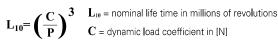
 $S_0 = 1$ for normal circumstances,

= 1.5 when vibrations are involved and

=2 - 2.5 for noiseless applications.

DYNAMIC LOAD CALCULATION:

For normal circumstances, the dynamic load coefficient is calculated and compared to the equivalent dynamic load to determine the theoretical service life of the bearing. The calculated service life is then compared to the standard for the application and industry.



When bearings run at constant speed, their life can be determined in working hours:

$$L_{10h} = \frac{1.000.000}{60n} \star \left(\frac{C}{P}\right)^3 \quad \begin{array}{ll} L_{\text{\tiny 10h}} = \text{life in hours.} \\ & n = \text{speed [rpm].} \end{array}$$

When the load on the bearing is strictly radial the equivalent dynamic load $P = F_{r}$.

When there's also an axial load involved the equivalent dynamic load is:

 $P=X*F_r+Y*F_a$

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BEARING AND SHAFT

BEARING

X radial dynamic factor;

Y axial dynamic factor;

The axial force only influences the equivalent dynamic load when Fa/ Fr > E

	CALCULATION				
		F _A /F	_R < E	F _A /F	_R > E
F _A /C ₀	E	Х	Υ	Х	Υ
0.025	0.22	1	0	0.56	2
0.04	0.24	1	0	0.56	1.8
0.07	0.27	1	0	0.56	1.6
0.13	0.31	1	0	0.56	1.4
0.25	0.37	1	0	0.56	1.2
0.5	0.44	1	0	0.56	1





THE AXIAL LOAD SHOULD NEVER EXCEED 20% OF THE DYNAMIC LOAD COEFFICIENT (C).

INDICATIVE SPEED FOR SHAFT TOLERANCES:

SHAFT DIAMETER				SHAFT TOLERANCE	S	
		h6	h7	h8	h9	h11
	Limiting speeds - Shaft tolerance					
mm	in.			RPM		
Ø12	-	9500	6000	4300	1500	950
Ø15	-	9500	6000	4300	1500	950
Ø16	-	9500	6000	4300	1500	950
Ø17	_	9500	6000	4300	1500	950
Ø20	-	8500	5300	3800	1300	850
Ø25	_	7000	4500	3200	1000	700
Ø30	-	6300	4000	2800	900	630
Ø35	-	5300	3400	2200	750	530
Ø40	-	4800	3000	1900	670	480
-	Ø1	7000	4500	3200	1000	700
-	Ø1 3/16	6300	4000	2800	900	630
-	Ø1 1/4	5600	3600	2500	800	550
-	Ø1 7/16	5300	3400	2200	750	530
-	Ø1 1/2	5000	3100	2000	700	500



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MAXIMUM RADIAL LOAD CAPACITY OF HOUSING:

Values indicated in the table below, are obtained at 23°C. Value indicated: N, Values of polyamide version (values of polypropylene version in brackets). For further information please consult System Plast® SmartGuide® pages regarding bearing type you are interested in.

-		

Dack

Series SQL ø mm				2
12-15-16-17	6500	(-)	6000	(-)
20	5800	(-)	5300	(-)

Series UCFH/UCFH-R ø mm	Q .)
12-15-16-17	6500	(-)	7000	(-)
20	7000	(-)	7500	(-)
25	7500	(-)	8000	(-)
30	8000	(-)	8800	(-)
35	8500	(-)	9000	(-)

Bearing m	ounting. Go to:
7 3 0	5 2 a

Series UCFL/UCFL-W ø mm			-	
20	6900	(3200)	7500	(3500)
25	6750	(2950)	7200	(3000)
30	12000	(5800)	13000	(6000)
35	11000	(5300)	12800	(6000)
40	10800	(5100)	12850	(5800)

Series UCFQ/UCFQ-R ø mm				
12-15-16-17	12000	(-)	12000	(-)
20	12500	(-)	11500	(-)
25	12500	(-)	11500	(-)
30	12800	(-)	11000	(-)
35	12300	(-)	10500	(-)
40	12000	(-)	10000	(-)

Series UCPA/UCPA-R ø mm						
20	6500	(3000)	2000	(1000)	4000	(1600)
25	7800	(3200)	2500	(1400)	4800	(1800)
30	11000	(4000)	2900	(1500)	5700	(2000)

Series UCT/UCT-R ø mm			6		Q	·
20	7800	(3200)	2500	(1400)	4800	(1800)
25	6500	(3000)	2000	(1000)	4000	(1600)
30	12000	(5500)	3500	(1800)	6000	(2400)
35	10000	(3600)	2900	(1500)	5500	(2000)

Series UCFG/UCFG-R ø mm			É	
20	12000	(-)	10000	(-)
25	12500	(-)	11000	(-)
30	12800	(-)	11500	(-)

Series UCHE/UCHE-R ø mm						
20	10000	(3600)	2500	(1400)	4800	(1800)
25	8500	(3200)	2000	(1000)	4000	(1600)
30	14000	(6500)	3500	(1800)	6000	(2400)
35	12000	(5500)	2900	(1500)	5500	(2000)

Series UCFA/UCFA-R ø mm						
20	8000	(-)	2000	(-)	2500	(-)
25	8500	(-)	3000	(-)	3000	(-)
30	9000	(-)	3500	(-)	3500	(-)





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STEEL CHAIN MATERIALS

MATERIALS



C45

CARBON STEEL: S AND SR

• Surface hardness: 44 HRC

C45 steel material allows a higher yield load capability than stainless steel but is not corrosion resistant, thus only suitable for dry environments. Oil lubrication may be necessary to reduce rust and drag. The through hardened plate material provides a uniform hardness and a high abrasion resistance.





- High mechanical strength
- · High abrasion resistance



STANDARD

STANDARD STAINLESS STEEL: SS AND SSL

AISI 430 Stainless steel material with good mechanical characteristics and corrosion resistance. An economical option for many conveying applications.

Lower load and lower wear resistance capability than our higher grade stainless steel materials.



FEATURES:

FEATURES:

· Standard stainless steel

BENEFITS:

BENEFITS:

- · Food grade
- Economical solution

EXTRA PLUS STAINLESS STEEL: SSE, SSSR, SSEL AND SPSL

High performance stainless steel, specially developed for high speed and heavy-duty application. Offers excellent corrosion resistance and surface hardness.



EXTRA PLUS





- Surface hardness of HRC 26-30
- Extremely flat and smooth surface finish
- High corrosion and wear resistance
- · Highest ultimate yield loading capability

BENEFITS:

- Food grade
- Typically used in glide liners and pressureless combiners and very long convevors
- · For improved product stability



AUSTENITIC STAINLESS STEEL: SSA

AISI 304 Austenitic stainless steel which offers high corrosion and acid resistance properties.





FEATURES:

• Stainless steel with 18% chrome and 8% nickel

BENEFITS:

- Food grade
- High corrosion and acid resistance material



"Z"	577	T.	T'S	RUST	Ø
H I GH	FOOD	MEDIUM HIGH	VERY HIGH	CORROSION	CHEMICAL
STRENGTH	GRADE	SPEED	SPEED	RESISTANCE	RESISTANCE



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MATERIALS

SPM

STANDARD PIN MATERIAL

FEATURES:

Special stainless steel with higher tensile strength and improved surface hardness.

These pins are offered as standard in most stainless steel and plastic chains.

· High wear, corrosion and acid resistance

BENEFITS:

PIN MATERIALS FOR STEEL AND PLASTIC CHAINS

· Longer wear life

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HB

SPECIAL PIN MATERIAL

Vacuum hardened stainless steel with exceptionally high wear resistance characteristics, good corrosion and chemical resistance, for high speed and or abrasive applications with steel chain.

Pin wear is the main contributing factor in chain elongation.

Due to their extraordinary hardness, HB pins offer a much better resistance against mechanical/abrasive wear and thus extended service life of chain.

HB (Harten Bolzen) pins are recommended for the following operating conditions and applications:

- Abrasive environment (e.g. crate and bottle conveyors)
- Short conveyors running at high speed (e.g. filler area, inspector sections, pressure less combiners)
- High load applications (e.g. accumulation area, full crate conveyors, full bottle accumulation tables, keg

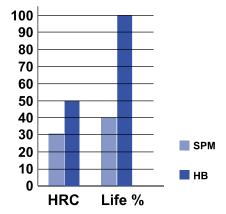
FEATURES:

• Hardness > HRC 48

BENEFITS:

- · Ultimate abrasion resistance
- · Outstanding wear life
- · Wear resistance twice of standard pin

SPECIAL PIN MATERIAL HB STANDARD PIN MATERIAL SPM





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PIN MATERIALS FOR STEEL AND PLASTIC CHAINS





PPM

PLASTIC PIN MATERIAL

Special reinforced acetal resin. For plastic chains only.



FEATURES:

· Optimum strength

BENEFITS:

- · Ultimate abrasion resistance
- Outstanding wear life
- Suitable for metal detection applications



FPM

FERRITIC STEEL PIN MATERIAL FOR MAGNETIC CHAINS

Ferritic Stainless-Steel pin is used for magnetic side-flexing chains. This material offers a great retention force in combination with magnetic curve.



FEATURES:

· High wear resistance

BENEFITS:

- Optimum retention force
- · Longer wear life



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PLASTIC CHAIN AND BELT MATERIALS



MATERIALS

LF

LFG

LOW FRICTION ACETAL RESIN

This material is commonly used in the market and offers an improved coefficient of friction. It is also suitable for use in high speed applications.

LFB

Plastic Chain Color: Light Brown (LF), Dark Grey (LFG) or White (LFW)

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Index chains. Go to:

LFW

Plastic Belt Color: Grey (LFG); Blue (LFB); white (LFW)

1 1 0 1 0 a



FEATURES: · Optimum strength and wear life **BENEFITS:** · Food grade





NG[®]evo

NG®evo conveyor components made from engineered plastic resin

Extra performance PBT with lowest coefficient of friction in our range, resulting in good strength and optimum wear resistance, reduced plate wear and reduced pitch elongation.

Available exclusively from Regal® System Plast® product,

Color: Grey Blue



FEATURES:

- Optimum strength and wear life
- Lowest coefficient of friction
- Optimum abrasive resistance
- Higher operating temperature in dry conditions

BENEFITS:

- Food grade
- High speed application
- Dry running application

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H I GH STRENGTH	FOOD GRADE	MEDIUM HIGH SPEED	VERY HIGH SPEED	VERY LOW COEFFICIENT OF FRICTION

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SPECIAL PLASTIC CHAIN AND BELT MATERIALS



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On request and for adequate quantities belts may be produced in other materials such as:



ANTI-STATIC ACETAL RESIN

Anti-static acetal material with improved surface conductivity for greater protection against static electrical discharges.



FEATURES: Antistatic

BENEFITS:

· Dissipates nuisance static build-up

Index belts.

continue



HT

HIGH TEMPERATURE RESISTANT

Polyamide material based for high temperature applications up to 140°C.

Color: Natural white

BENEFITS:

- · High temperature resistance
- Food grade

XT

EXTRA TEMPERATURE

This material with fiber glass-reinforced offers an excellent heat and chemical resistance for belt/chain and a good load capacity in respect to other materials. XT is suitable for extra high temperature applications up to 240° C.

Color: Beige

BENEFITS:

• Very high temperature up to 240°C

PP

PPW

CHEMICAL RESISTANCE

For belts: polypropylene material for greater chemical and temperature resistance. Polypropylene has lower mechanical strength than acetal. For belts (PP) the pins and clips are also made of Polypropylene.

Color: Beige (PP), White (PPW).





BENEFITS:

- · Food grade
- · Higher temperature resistance
- Optimum chemical resistance

PPG

PPLG

HIGH STABILIZED HOMOPOLYMER

For belts: Polypropylene material for a greater chemical and temperature resistance. This Polypropylene is a high stabilized homopolymer, combining high impact strength and stiffness. It has been developed especially for high detergent resistance and heat stability. Apart from pasteurizers, this material is also used in industrial washing machines, heat exchangers and dishwashers.

Color: Dark Grey (PPG), Light Grey (PPLG).

BENEFITS:

- · Food grade
- · Higher temperature resistance
- · Optimum chemical resistance

CR

For chains: reinforced polypropylene material for greater acid and chemical resistance. Polypropylene has lower mechanical strength than acetal. Reinforced PP is not food grade.

Color: White (CR, for chains only)



Special colours

On request and for adequate quantities, chains and belts may be produced in other colours.

Storage of plastic chains and

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FOOD GRADE	ANTISTATIC	HIGH TEMPERATURE RESISTANCE	VERY HIGH TEMPERATURE RESISTANCE	CHEM I CAL RES I STANCE



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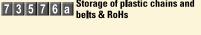
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Measurement procedure





SPECIAL PLASTIC CHAIN AND BELT MATERIALS



MATERIALS

PP-FR-EC

FLAME RETARDANT, ELECTRICALLY CONDUCTIVE POLYPROPYLENE





Flammability according to UL94 is classified as V0. Due to the very low surface resistivity, this material can be considered electrically conductive.

Colour: Black.



Special colours

On request and for adequate quantities, chains and belts may be produced in other colours.



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PIN MATERIALS FOR BELTS







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PBT

PBT

Most commonly used pin material in System Plast® belts. This material offers a high abrasion resistance as well as a low noise operation.

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POM

POM

Polyacetal pins are used in System Plast® belts. This pin material gives optimum strength.

PP

PP

Polypropylene pins are used in all belts made of PP material. The pin material is adapted to the high temperature and high chemical resistance of the belt material.

SS

SS

Ferritic stainless steel pin is used for side-flexing belt 2120M. This material offers a great retention force in combination with magnetic curve.



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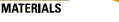
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RUBBER MATERIALS







TPR

TPR (SEBS)

TPR is used for VG chains and belts and for plastic gripper chains. TPR is a SEBS type rubber, which assures an optimum bonding on the plastic base material.

Mechanical properties data available on MDS.

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NBR

NBR

NBR rubber is used for our gripper chains serie 1874.

It offers a soft grip and a good resistance against oils.

EPDM

EPDM

EPDM rubber is used for our gripper chains.

It offers good weather and chemical resistance, although contact with oils, gasoline and concentrated acids must be avoided.

EPDM-PP

EPDM-PP

EPDM-PP rubber is used for our gripper chains.

It offers improved chemical resistance and can be used at higher temperatures.

Resistance against steam is good.



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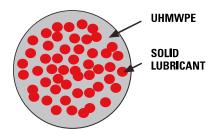
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MATERIALS

NOLU®-S

WHAT ARE NOLU PLASTICS?

Nolu® plastic parts are made with Regal's exclusive compounds of UHMWPE and solid lubricants/unique ingredients. They provide reduced coefficients of friction while maintaining the characteristics of UHMWPE. Our featured component brands are Nolu-S, Nolu-SR and Nolu-SX.





NOLU®-S

Nolu-S parts are made with a blend of UHMWPE and other dry lubricants that maintain good wear characteristics while significantly reducing coefficient of friction. Its unique self-lubricating properties make it ideal for applications requiring reduced friction and noise reduction.



FEATURES:

- · RAM extruded or machined
- Very low coefficient of friction material flow
- High-impact strength
- Chemical resistant with minimal moisture absorption
- The Nolu®-S parts ingredients comply with EU and FDA food regulations
- Noise suppressant
- Easy to machine most general power tools are sufficient

BENEFITS:

- Extends the life of mating surfaces
- Reduce surface marring and damage
- Reduces energy consumption
- Minimizes product pulsation on conveyors
- Low noise operation



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NOLU®-SR PARTS

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Nolu-SR parts have the same unique self-lubricating properties as Nolu®-S parts, however, they are made with additional proprietary ingredients. These unique ingredients increase the hardness of the material, providing superior wear resistance as compared to standard UHMWPE. Benefits of Nolu-SR parts include high wear resistance, extended wear life, minimization of product pulsation and low noise.





FEATURES:

- RAM extruded or machined
- Superior wear resistance greater than standard virgin or repro material
- · Low coefficient of friction promotes material flow
- High-impact strength
- The Nolu-SR parts are made with ingredients that comply with EU and FDA regulations
- Chemical resistant with minimal moisture absorption
- · Easy to machine most general power tools are sufficient

BENEFITS:

- Even greater wear resistance
- Extends the life of mating surfaces
- · Reduce surface marring and damage
- Reduces conveyor chain loads
- Reduces energy consumption
- · Reduces product pulsation on conveyors
- Low noise operation
- Suitable for running dry at higher speed

NOLU®-SX PARTS



Nolu-SX parts has the same unique self-lubricating properties as Nolu-SR parts; however, Nolu-SX parts has a material blend that reduces the thermal expansion characteristics of the material. The thermal expansion is reduced up to 40% compared to standard UHMWPE products, making it the ideal material for neck guides.

Nolu plastics should be stored at room temperature between 50°F (10°C) and 104°F (40°C) to prevent premature aging of the material. They should also be stored in a manner to prevent distortion. Regular cleaning is also recommended with warm water and soap agents with a pH value of 4.5 to 9.0. For more technical information regarding Nolu parts, please contact our Application Engineering Group.



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MATERIALS

CURVE, WEARSTRIPS, SIDE GUIDE, SPROCKETS AND RETURN ROLLER

CURVES, WEARSTRIPS AND SIDE GUIDE MATERIALS

UHMWPE GREEN

ULTRA-HIGH MOLECULAR WEIGHT POLYETHYLENE (UHMWPE)

UHMWPE BLACK

This Polyethylene polymer with extremely high molecular weight has several unique properties such as high abrasion resistance, impact strength and low coefficient of friction. The UHMWPE are used for our side guide profiles, curves and wearstrips.

UHMWPE WHITE

UHMWPE WATER BLUE

UHMWPE-AS BLACK

ANTISTATIC ULTRA-HIGH MOLECULAR WEIGHT POLYETHYLENE (UHMWPE-AS)

This PE polymer with extremely high molecular weight has several unique properties such as good wear resistance, good impact strength, good sliding and antistatic properties. These properties make the material suitable for curves and straight tracks.

Surface Resistivity: 10^3 - $10^4 \Omega$; Volume Resistivity: 10^3 - $10^4 \Omega$ cm.

NOLU®-CP PARTS

Nolu-CP parts are made with Regal's exclusive compounds of UHMWPE and micro glass beads. Curves and neck guide made with this material are the most suitable for abrasive working conditions.

POM - GREEN

POM-ACETAL

POM - WHITE

This polyacetal used for roller guide provide a high rigidity, hardness and toughness and good chemical resistance to alkalis.

SPROCKETS, IDLER MATERIALS

PA BLACK

GLASS FIBER REINFORCED POLYAMIDE (PA-FV)

The reinforced polyamide with glass fiber is used for molded sprockets and idlers to offer a high strength and wear resistance.

PA WHITE

POLYAMIDE (PA)

The cast polyamide 6 is used for machined sprockets and idlers to offer a high strength and wear resistance.

RETURN ROLLERS MATERIALS

PE-HD BLACK

HIGH-DENSITY POLYETHYLENE (PE-HD)

PE-HD GREEN

The high-density polyethylene is used for return roller and it has a good abrasion resistance, impact strength and a low coefficient of friction.

PA BLACK

POLYAMIDE (PA)

PA GREEN

The impact polyamide resin is used for return roller to offer a high strength and wear resistance.



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MATERIALS

STORAGE OF PLASTIC CHAINS AND BELTS AND RoHS

STORAGE OF PLASTIC CHAINS AND BELTS

Plastic chains, belts and components offer best stability and resistance against environmental effects at appropriate storage:

- In the original packaging, and not exposed to the environment
- Without environmental radiation / UV light / direct sunlight
- Dry
- In a non-aggressive environment
- A consistent temperature between 10°C (50° F) and 40°C (104° F)

FIRST IN, FIRST OUT

- We have applied that procedure in our logistic department.
- · We recommend this procedure to any external warehouse.

Do not stack pallets or other heavy goods on top of chain packs. Chains inside the packs might get damaged. Do not stack chain packs higher than the original stacking height - as dispatched from our shipping department.

PRODUCT SHELF LIFE

PRODUCT CLASS	TIME FROM MANUFACTURING DATA	REFURBISH (INSPECT, CLEAN, RELUBRICATE AND REPACKAGE)	DON'T USE
All solid steel and cast iron	5 years	5 years	10 years
Plastic: rubber and EPDM	3 years	Not Applicable	3 years
Plastic: UHMWPE	3 years	Not Applicable	3 years
Plastic belts and Chains	3 years	Not Applicable	3 years
Plastic idler and sprockets	3 years	Not Applicable	3 years
Bearings	3 years	3 years	6 years

RoHS

System Plast® CHAINS, BELTS AND COMPONENTS materials do meet the base requirements of this european directive. Therefore, the RoHS (Restriction of Hazardous Substances) directive of 2002/95/CE and subsequent modifications do not apply to our products.

ATEX

FOR ATEX APPLICATION, CONTACT OUR TECHNICAL SUPPORT.

REACH

Regal materials meet the requirements of this European regulation.



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DISPOSAL OF PRODUCTS





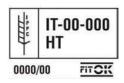
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DISPOSAL OF PLASTIC CHAINS AND BELTS

The non-electric nor electronics waste (plastic, rubber and metals parts) are not considered special waste and must be disposed of in strict compliance with the local regulations regarding non-special waste.

DISPOSAL OF MODSORT® MODULES

The wood packaging material of Modsort® modules is fully recyclable and FITOK certified in accordance with the ISPM-15 regulation.



The electrical and electronic parts of Modsort® modules listed below are marked with the symbol



and classified as Waste of electric and electronic equipment (WEEE) in accordance with the European Directive 2012/19 / EU on WEEE.

- Control Card
- Divert Belt and Roller Top Belt's Motorized Drive Rollers (MDRs)
- Photo eyes
- Power Supply Unit

These WEEE must be disposed correctly at a suitable collection point according to the procedures in use in the country of disposal.



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COEFFICIENTS OF FRICTION OF STANDARD MATERIALS

Below listed coefficients can be used as a guideline. Depending on environmental and application requirements, (temperatures, lubricant, material combinations, dirt/debris, product and chain/belt surfaces, etc.) the coefficients are subject to some variation. Values are generally conservative.

COEFFICIENT OF FRICTION BETWEEN CHAIN/BELT AND WEARSTRIP:

			WEAR STRIP MATERIAL						
CHAIN/BELT MATERIAL	LUBRICATION	STAINLESS STEEL STEEL	UHMWPE PA	NOLU®-S PARTS	WOOD	RETURN ROLLER	EXTRA WITH METAL STRIP		
	Dry	N/A	0.35	0.32	0.35	0.10			
Stainless steel	Dry lube	0.39	0.26	0.23	N/A	0.08	not applicable		
	Water&Soap	0.20	0.18	0.15	N/A	0.05			
Carbon steel	Dry lube or Oil lube	0.39	0.26	0.23	N/A	0.08	not applicable		
	Dry	0.24	0.20	0.18	0.23	0.10	0.19		
LF - Acetal	Dry lube	0.18	0.17	0.13	N/A	0.07	0.14		
	Water&Soap	0.15	0.10	0.10	N/A	0.05	0.10		
NG®evo	Dry	0.20	0.16	0.13	0.17	0.10	0.14		
conveyor components made from engineered	Dry lube	0.16	0.10	0.08	N/A	0.07	0.09		
plastic resin	Water&Soap	0.14	0.09	0.08	N/A	0.05	0.09		
DD / CD	Dry	0.29	0.24	0.21	0.22	0.10	0.23		
PP / CR	Dry lube	0.22	0.18	0.16	N/A	0.07	0.17		

Valid for ambient temperature 21°C (70° F)

COEFFICIENT OF FRICTION BETWEEN CHAIN/BELT AND PRODUCT:

CHAIN/BELT				PRODUCT	MATERIAL		
MATERIAL	LUBRICATION	PAPER CARTON	METAL (STEEL)	ALUMINUM	PLASTICS INCL. PET	GLASS (RETURN)	NEW GLASS, CERAMICS
	Dry	0.40	0.50	0.35	0.30	0.47	0.35
Stainless steel	Dry lube	N/A	0.34	0.29	0.24	0.30	0.29
	Water&Soap	N/A	0.20	0.15	0.15	0.21	0.15
Carbon steel	Dry lube or Oil lube	N/A	0.34	0.29	0.24	0.30	0.29
	Dry	0.40	0.45	0.32	0.27	0.40	0.29
Speed - Line	Dry lube	N/A	0.29	0.26	0.22	0.25	0.23
	Water&Soap	N/A	0.15	0.14	0.13	0.18	0.13
	Dry	0.28	0.25	0.25	0.21	0.24	0.20
LF - Acetal	Dry lube	N/A	0.19	0.17	0.15	0.17	0.14
	Water&Soap	N/A	0.15	0.14	0.13	0.14	0.12
NG®evo	Dry	0.20	0.18	0.15	0.13	0.14	0.12
conveyor components made from engineered	Dry lube	N/A	0.15	0.13	0.11	0.12	0.11
plastic resin	Water&Soap	N/A	0.13	0.12	0.10	0.11	0.10
LBP	Dry	0.10	0.10	0.10	0.10	0.10	0.10
VG	Dry	0.60	0.73	0.50	0.50	0.50	0.50
PP / CR	Dry	0.40	0.30	0.32	0.28	0.29	0.26
FF/Ch	Dry lube	N/A	0.23	0.25	0.21	0.22	0.20

Valid for ambient temperature 21°C (70° F)

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COEFFICIENT OF FRICTION - MEASUREMENT PROCEDURE



MATERIA

In this section, we will describe the measurement method for coefficient of friction (COF or µ) between belt/chain and product on a running conveyor.



For iCOF® device.

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The COF is defined as the ratio between the force necessary to move one surface horizontally over another and the pressure between the two surfaces. It can be measured by dividing the force required to push or pull an object by its weight. The pull and the product weight can be determined by measurement using a spring or force gauge. Please follow the below instructions, ensuring both conveyor and sample product are clean and clear of all debris:

1- At first be sure to have:

- a. Mechanical or digital spring gauge with suitable sensitivity and accuracy
- b. A spirit level gauge (in case of inclined conveyor)
- c. A suitable sample of product weight (bottles/cans etc.)
- d. Available space on the running conveyor where the COF measurement is required

2- Measure the sample weight (W):

- a. Use sample product with suitable weight. It is recommended to collect one or more products to get 1 kg of product weight.
- **b.** Measure the product weight using a scale or spring gauge
- c. Record the value when the scale is stabilized

3- Measure the pull (F_0) :

- a. Be sure that conveyor is running and the surface is level. Otherwise measure carefully the angle
- b. With the same sample as step 2.a), take measurements in the same location on the conveyor and at the same speed if possible
- c. Secure the gauge to a fixed point on the conveyor or hold the gauge by hand avoiding movements. If possible, steady your hand by resting it on conveyor frame or guide rail
- d. Ensure that the gauge is parallel to the conveyor
- **e.** Put the sample product on the conveyor, attach to spring gauge, and measure the horizontal pull (\mathbf{F}_0) .
- f. Take the value when the measurement is stabilized
- g. Make three (03) measurements at the same conditions
- h. Make the average of the obtained three (03) values
- i. Repeat steps 2 and 3 for multiple different containers

4- COF calculation:

- **a.** Be sure that the average W and F_{θ} values are in the same measurement unit (g or kg or lb);
- **b.** Use the formula $\mu = \frac{F_0}{W} = \frac{\text{measured pull}}{\text{measured weight}}$ to obtain the average value of COF; (4b)

Be aware that for some gauges it is required to compensate the reading with the plunger weight (P) (1) since it is used horizontally. Refer to page 7 3 5 8 1 a for further details.

With the plunger weight compensation, the formula 4b) becomes:

$$\mu = \frac{(F_0 + P)}{W} \qquad \qquad \frac{\text{measured pull} + \text{plunger weight}}{\text{measured weight}}$$

If the conveyor is inclined with respect to the running direction, the angle α is to be considered as following:

- Measure the angle α (usually from 2° to 5°);
- Use the formula $\mu = \frac{(F_0 + P)}{W cos\alpha} tan \alpha$ (Running direction downwards) (1).
- Use the formula $\mu = \frac{(F_0 + P)}{W \; cos \alpha} + \; tan \; \alpha \;$ (Running direction upwards) (1).

Notes: (1) P=0 for spring gauges with adjustable preload. Refer to good measurement practice for further details.



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COEFFICIENT OF FRICTION - GOOD MEASUREMENT PRACTICE









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MATERIALS

Here we describe good practices when measuring the COF within belt/chain in a running conveyor.

SELECTION OF THE INSTRUMENT

The higher the sensibility, the more accurate the measurement. 10 g divisions and +/- 10% (or ounces) is a good compromise for a rough estimation.

A. Gauge with mechanical spring (0-10 kg)

- i. Needs certain weight
- ii. Needs to compensate spring preload and "plunger"
- iii. Is more accurate / repeatable

There are spring gauges on the market with adjustable preload, in order to compensate the plunger. For those types of gauges, just adjust the reading scale to 0 when reading in horizontal position. Remember to reset the scale to 0 when reading values in vertical position (i.e. when weighting a bottle).

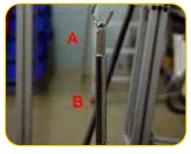
B. Digital gauge (0-45 kg)

- i. Needs lower weight
- ii. Don't always need to compensate for the "plunger"
- iii. Less accurate / repeatable

REQUIRED ACTIVITIES AND CHECKS PRIOR TO MEASUREMENT

A. Gauge with mechanical spring: Plunger preload determination

- i. Prior to the measurement ensure that also the spring dynamometer reads zero when in vertical working position. Tare if necessary.
- ii. Weight the dynamometer on a balance or using a second scale. Record the value. Weight reading of the total scale x = A plunger preload + B scale body [g]
- iii. Hang the dynamometer upside down. If the instrument's own weight is not enough to pull the plunger by at least 10% of his own full scale, hang some additional weight at the bottom. The additional weight should be deducted from the reading.
- iv. Weight reading of the scale upside-down y = B scale body A plunger [g]. Read and store the value on the scale.
- Plunger weight (spring preload) will be . A = $\frac{x y}{2}$ RECORD THIS VALUE!



Example:

Gauge weights 140 g, upside-down reading is 36 g, the spring-preload is (140-36)/2 = 52 g. For the horizontal reading, you will have to add this value to the displayed forc.



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COEFFICIENT OF FRICTION - GOOD MEASUREMENT PRACTICE

B. Digital gauge: Plunger preload determination

If your digital tool offers the possibility to switch it to horizontal, you don't need to calculate the plunger preload. In fact, in horizontal configuration, the cell automatically pre-sets to zero.

Some digital tools, on the contrary, do not allow to be switched to horizontal (they need a certain weight, given by the plunger when lying vertically to activate the cell). Under such condition proceed in plunger preload determination as per previous point.

Periodically check your instrument and compare the measurements with a second one. This is required to check any possible deviation of measurements due to loss of tare.

Recommendations

- A higher is the sample product weight, a lower is the error in the estimated value
 - If possible, we recommend using multiple containers to increase product weight
- · Measure the product weight close to the ground so as to prevent any dropped bottles from breaking
- · Take a stabilized measurement. Instrument gauge has a response time, so we recommend respecting this time to read an accurate value
- The more measurements that are taken and averaged, the more reliable the result will be.
- · A smooth, clean surface allows an accurate COF measurement without the influence of wear
- Using the same sample(s) in all measurements will lead to a uniform and repeatable value
- · Securing the gauge to a fixed point on the conveyor will remove human error from holding the spring gauge
- Ensure you hold the gauge and the plunger parallel to the conveyor, so as to achieve an accurate measurement



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CHEMICAL RESISTANCE OF STANDARD MATERIALS

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Data shown in the table was taken from laboratory tests performed on unstrained samples and are merely indicative.

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Chemical resistance under normal working conditions can depend on various factors, such as stress and temperature, concentration of the chemical agent and contact time. Conveyor rinsing can also be a huge factor, as a corrosive chemical that is allowed to dry and concentrate will be more problematic than one that is allowed contact for a short time and thoroughly rinsed away.

VALID FOR AMBIENT TEMPERATURE (21°C / 70°F)

				MET	ALS								PLAS	TICS					RUBBERS							
CHEMICAL AGENT	EX	ΓRA	AIS	304	AIS	316	OT.	NI	LF (P	OM)	NGE ((PBT)	Р	Р	P.	A	Р	E	EPI	DM	NE	BR	SE	BS	VITO	N*
	C%		C%		C%		C%		C%		C%		C%		C%		C%		C%		C%		C%		C%	
Acetic Acid	5	\	20	V	100	√	T	0	5	×	10	~	40	✓	10	×	10	✓	25	✓	T	×	25	0	20	×
Acetone	1	✓	25	✓	1	✓	1	✓		0		0	Ī	✓	100	✓		✓	1	1		×		0	T	×
Acrylonitrile	1	T			T	1	1						T	√	100	✓		✓	1	1		×	1	0	T	×
Aluminium chloride	<u>-</u>	·		0	10	0	T						†	0	10	✓		✓	İ	√		✓	<u></u>	✓	SA	V
Aluminium sulphate	<u> </u>	·			SA	✓	T							1	10	✓	SA	✓	İ	1		✓		1	SA	1
Amyl alcohol				✓		✓	1					1		✓	10	✓		✓	Ì	✓				✓	1	✓
Ammonia		✓	100	✓	T	1	1	×		✓		0	30	✓	10	✓		✓	İ	✓		0	ļ	0		0
Ammonium chloride	1	T	1	0	T	✓	1						10	✓	10	✓		✓	İ	✓		✓	1	✓	SA	√
Aniline	····	✓	T	V	T	1	T						†	√	100	0	3	✓	İ	×		×		×	Ť	V
Barium chloride		ļ	1	0	SA	1	T		İ			İ	†	1	10	✓	<u> </u>	✓	İ	1		✓	<u> </u>	✓	T	1
Beer	t	✓		V		1	T	√		✓			 	1		1		✓	İ	1		✓		✓	T	1
Benzene	<u> </u>	✓	70	0		1	T		İ	✓		1	†	1			İ	0	†	×			İ	×	·	·
Benzoic acid	····	·	100	V	SA	V	T		İ			/	SA	V	SA	0	İ	✓	†	×		√	†	×	†	/
Benzol		†	+	V	†	1	†	√		√		/	†	0	100	√		√	†	×		×	†	×	†	0
Boric acid		0	SA	1	†	1	†			×	10	1	SA	1	10	1	SA	√	†	1		√	†	1	SA	1
Brine	10	×		0	†	1	†					1	†	0		0	†	√	†	1			†	0	†	
Butter	<u>†</u>	†	+	V	†	1	†	·····		√		/	†	1		1		√	†	V		√		0	†	V
Butyl acetate	····	†	İ		†	1	†			√		0	†	0	100	/	†	√	†	0	·		†	0	†	×
Butyl alcohol		†	†	1	†		†		†	√		1	†	1	100	1	†	√	†	1	†	0	†	1	†	1
Butyl glycole		†	İ		†	1	†		†				†	1	100	1		√	†	1	†		†	1	†	·
Calcium chloride		×	·	0	†	1	†	√	10	√		1	50	1	10	/	SA	√	†	1		√		√	SA	1
Carbon sulphide			+	V	 	V				√				V	100	√		0	t	×		×		×		/
Carbon tetrachloride			10	· ·				√		√		· · · · · · · · · · · · · · · · · · ·	 	×		√			t			×			 	/
Chlorine water	t	×		×	†	0	+			×		×	†	×				√	3	0			3	0	†	
Chloroform	····t	0	10	· ·	†	1	†	√		×		×	†	0	100	×		×	İ	×		×		×	†	/
Chromic acid	····t	·	25	· ·	50	0	†			×		0	†		1	0	50	0	50	0		×	50	×	50	/
Citric acid	10	· ·	·	· ·	SA	1	†	×		0	10	/	10	V	10	0	SA	√	†	1		√		V	SA	/
Cyclohexane		†	†		†	1	†		†			1	†	1	100	1	†	/	†	×	†	√	†	×	†	1
Cycloexanol			†		 	1						/	 	1	100	1		√	t	×		√	·	0	 	1
Decalin			†		 	1						0	 	0		/			t	×		0		×	 	×
Dioxane			†			V	†			0		√	†	0		√	†	√	t	0		×	†	×	†	·
Distilled water		√	10	/	†	1	†	√	·	<i>\</i>		/	 	1		<i></i>	†	√	t	7			 	✓	†	×
Ethyl acetate	····t	†	†	0	†	1	†	·····		0	·	0	t	1	100	1	†	√	t	t	†	×	t		†	0
Ethyl alcohol	····t	t	+		t		†	·····	·	√ · ·		<u>-</u>	96	· · ·	96	· ·	96	√		·	†	0	t		t	<i>\</i>
Ethyl chloride	····t	t	+	· ·	t		†		·	√			†	×	100	· ·	†	0	t	·	†	0	t		t	×
Ethyl ether	····-t·····	t	†	t	t	· ·	t	<u>-</u>	·	·····		· · · · · · · · · · · · · · · · · · ·	t	V	100	· · ·	†	0	t	·	t	<u>-</u>	t	·	t	
Ferric chloride		t	+	0	t	· ·	 	L	†	0	10	····	 	· ·	10	· ·	†		t	· · ·	†	✓	 	· ·	SA	· ·
Food fats			100	- V	t	· ·	 	L	†	- V	 :×	· · ·	 	·	† <u></u>	· ·	 	· · ·	t	0	 	· · ·	 	0	†	· ·
Food oils		·	1.00	· ·	 	· ·	t	ļ	 	· /		· ·	 	-	 	· ·	 	· ·	ł	 	 	· ·	 	 	 	· ·

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ABBREVIATION

O = fairly good resistance depending on use conditions **SA** = saturated

x = insufficient resistance (not recommended) **blank spaces** = no tests performed

All values are only applicable to temperatures below 21°C / 70°F.

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VALID FOR AMBIENT TEMPERATURE (21°C / 70°F)

				MET	TALS								PLAS	TICS								RUB	BERS			
CHEMICAL AGENT	EXT	ΓRA	AIS	304	AIS	316	OT.	.NI	LF (P	(MO	NGE	(PBT)	P	P	P.	A	Р	E	EPI	DM	NE	3R	SE	BS	VIT	DN*
	C%		C%		C%		C%		C%		C%		C%		C%		C%		C%		C%		C%		C%	
Formaldehyde	T	V	T	✓	T	✓	T	V	T	V	T	V	40	✓	30	✓	T	0	T	0	T	0	T	0	40	×
Formic acid	2	0	İ	×	100	1	†	✓	10	×		0	İ		10	×	85	1	İ	✓	†	×	T	✓	T	0
Freon 12	1	ļ		✓	1		T					✓				✓	100	0	1	ļ		✓	Ī		1	✓
Fresh water	1	✓		✓	T	✓	T			√		1		✓		✓		1	1	✓		✓	Ī	✓	1	✓
Fruit juice	1	✓		0	T	✓	T			✓		✓		✓		✓		✓	1	✓		✓	Ī	✓	1	✓
Gasoline	1	√		✓		√	Ť	0		✓		0		0		√		0	†	×		0		×	†	✓
Glycerine	1	√		✓	1	1	Ī	√		✓		1		✓		✓	T	1	İ	✓		✓	Ī	✓	1	✓
Hydrocloric acid	1	×		×	1	×	Ī	0	35	×	20	0	30	✓		×	35	✓	15	✓		0	15	✓	37	✓
Hydrofluoric acid	1			×	1	×	Ī				5	0	40	✓		×	70	1	İ		Ī	×	Ī		48	✓
Hydrogen peroxide	3	√		✓	100	√	Ī		30	✓	5	✓	1			×	10	0	30	0	Ī	×	30	×	90	✓
Isopropyl alcohol	1	ļ		Ī		1	1			✓		1		✓		✓			1	1			1	✓	1	✓
Lactic acid	1	0		†	<u> </u>	1	Ť	×		✓	10	1	20	✓		✓		1	İ	0		✓	<u> </u>	0	Ť	✓
Linseed oil	1			V	<u></u>	1	Ť	İ		✓		1	†	✓		√		1	İ	0		√	<u> </u>	×	Ť	
Magnesium chloride	1			0		✓	Ť					√		✓		√		√	İ	√		√		✓	SA	✓
Methyl acetate	1			0		✓	Ť			0		0		✓		√		√	İ	0		×		×		×
Methyl alcohol	1		80	V		1	Ť	1		√		1				1		√	İ	1		0		√		0
Methylene chloride	1	0		0		1	Ť			×		×	1	0		1	·	0	İ	×		×	<u> </u>	×	1	0
Milk	1	V		V		1	Ť	1		V		1	1	✓		1	T	1	İ	0		1	<u> </u>	1	1	V
Mineral oil	1		T	V		1	Ť	·		V		/	1	✓		1	T	1	İ	×		1	<u> </u>	×	1	/
Nitric acid	25	0	65	✓	<u> </u>		Ť	İ		×			†	✓		×		0	İ		10	×	<u> </u>		70	✓
Nitrobenzene	1					1	Ť	İ				1	†	✓		0		1	İ	×		×	<u> </u>	0	†	0
Oxalic acid	1		65	V	<u></u>	1	Ť	İ			10	1	†	✓		0		1	İ	0		0	<u> </u>	0	Ť	✓
Paraffin	1					✓	Ť	İ		✓		✓						√	İ	0				×	+	
Petroleum	1			✓		√	Ť	√		✓		√		✓		√		×	†	×		√		×	+	✓
Petroleum ether	1			✓	1	1	Ī	√		✓		0		✓		✓	T	1		×		×	Ī	×	1	✓
Phenol	1			✓	1	1	Ī			0		×	1	✓		×	T	✓		0		×	Ī	0	1	✓
Phosphoric acid	25	0		×	1	✓	Ī	×		×		×	1	✓		×	T	1	1	✓	20	0	Ī	✓	85	✓
Potassium bichromate	1			ļ	SA	✓	Ī					0		✓		0	SA	1	İ	✓	Ī	0	Ī	0	SA	✓
Potassium bromite	1	ļ		Ī		✓	T					✓		✓		✓	10	1	1	✓		✓	Ī	✓	1	✓
Potassium hydroxide	1	✓	50	✓	1	1	T			✓		×		✓		✓	T	✓	1	1		0	1	✓	1	✓
Potassium permanganate	1			✓	<u></u>	√	Ť	İ	10	0		√	†	✓		×		✓	10	√		×	10	0	Ť	✓
Sea water	1	×		✓		✓	Ť	√		0		✓		✓		√		√	İ	√		√		0	†	✓
Silicone oil	1	İ	İ	†	T	1	†	T	İ	İ	İ	1	†	√	İ	1	T	1	t	1	†	1	T	√	Ť	√
Silver nitrate	1	İ		0	1	1	1	Ī	1	·	·		1	✓		1	T	1	İ	Ī		0	1		1	✓
Sodium carbonate	1	√	100	V	SA	1	1	Ī	10	V	10	1	1	✓		1	1	√	Ì	1		1	1	1	1	✓
Sodium chloride	1	0	<u> </u>	0	T	1	Ť	1	1	V	10	1	İ	✓	<u> </u>	1	T	1	İ	1	T	1	Ť	1	SA	V
Sodium hydroxide	40	1	İ	V	60	/	†	İ	İ	√	10	×	†		İ	1	†	1	†	1	İ	0	T	✓	†	T
Sodium hypochlorite	1	İ	İ	×	SA	0	†	ļ	İ	×	10	1	†	✓	İ	1	†	1	10	1	†	×	10	0	5	V
Sodium si l icate	1	İ	100	V	†	1	†	İ	İ	İ	İ	İ	†		†	✓	†	1	t	1	T	√	†	1	†	✓

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√ = good resistance C = concentration

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CHEMICAL RESISTANCE OF STANDARD MATERIALS

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VALID FOR AMBIENT TEMPERATURE (21°C / 70°F)

				MET	ALS								PLAS	TICS								RUBI	BERS			
CHEMICAL AGENT	EXT	TRA	AISI	304	AISI	316	OT.	NI	LF (P	OM)	NGE (PBT)	PI	P	P	A	P	E	EPI	DM	NB	R	SE	BS	VITO	*NC
	C%		C%		C%		C%		C%		C%		C%		C%		C%		C%		C%		C%		C%	
Sodium sulphate	T	[100	✓	T	√	T		T	Ī	[T	✓	SA	✓	T	0	T	√	T	√	T	✓
Soft drinks				✓		✓				✓		✓		✓		✓		✓	1	✓		✓		✓	Ī	✓
Suds		ļ		✓		✓				✓	10	✓		✓		✓		✓	1	✓		✓		1	Ī	1
Sulphuric acid		×		×		0	1	✓		×	2	✓		✓		×	80	✓	50	✓		×	50	0	95	1
Tartaric acid		✓	50	✓		✓	1	×		0	50	✓		✓		✓		✓	1	0		✓		✓	T	✓
Tetrahydrofuran	<u> </u>					✓				0		✓		0		✓		0	İ	×		×		×		×
Tetralin	1			×		✓	1			✓		✓		×		✓	1	✓	1	×		×	İ	×	İ	✓
Tincture of iodine	<u> </u>			0		✓		×						✓	T	×	1	✓	İ	0	1	×		0	†	1
Toluol	1	✓				✓	1					✓		✓	1	✓	1		1	×		×		×	T	0
Transformer oil	···	1				✓	T			✓		√		0	T	✓	1	✓	1	×	T	✓		×	<u> </u>	1
Tricholoethylene				×	100	✓	T		1	0		×		0	T	0	1	0	İ	×	T	×		×	†	1
Triethano l amin	<u> </u>					✓	T					✓		✓	T	✓	1	✓	İ	0	T	×		0	†	×
Turpentine	<u> </u>	V		✓		✓	T			×		✓					1	0	İ	×	T			×	†	
Vase l ine	<u> </u>					✓						✓				✓	1	0	1	×	T	✓	İ	×		✓
Vegetable juice	 	✓		✓		✓				√		✓		✓		✓	1	✓	1	√		✓	İ	√		✓
Vegetable oils		1		✓		✓	1			✓		×		✓		✓	1	✓	1	0		✓		0	İ	✓
Vinegar		V		√	100	√		√		√	10	√		✓		√	1	✓	25	√		0	25	0		×
Water and soap		✓		✓		✓				✓		✓		✓		✓		✓		✓		✓		✓	T	✓
Whisky		✓		✓		✓	Ι	✓		✓		✓		✓		✓	I	✓		✓		✓		✓	Ţ	✓
Wine		✓		✓		✓		✓		✓		✓		✓		✓		0		✓		✓		✓		✓
Xilol		✓		✓		✓		0		×		✓		×		✓		✓	1	×		×		×		✓

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Storage of plastic chains and belts & RoHs

Coefficient of friction of 7 3 5 7 8 a standard materials





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APPLICATION TEMPERATURES



MATERIALS

For low and high temperature application please consult our Application Engineering department.

Be aware:

- If a chain, belt or other component is composed of several materials, all materials will have to be considered individually.
- Please note that the properties of a material change when the temperature changes
- In general, the impact resistance is reduced at low temperatures.
- The strength and wear resistance are reduced at elevated temperatures.
- High temperatures can also influence chemical resistance
- Some materials absorb moisture and expand/swell

	OVERDOL		MIN	MAX	. TEMP. °C	MIN	MAX	. TEMP. °F
MATERIAL	SYMBOL	USER FOR	TEMP. °C	DRY	WET	TEMP. °F	DRY	WET
Carbon steel	C45	Steel chains, roller chains	- 70	180	not recommended	- 95	350	not recommended
Ferritic stainless steel	Standard	Steel chains	-30	400	130	- 20	750	265
Extra stainless steel	Extra	Steel chains	-30	400	130	- 20	750	265
Extra plus stainless steel	Extra plus	Steel chains	-30	400	130	- 20	750	265
Austenitic stainless steel	Austic	Steel chains, roller chains	-30	400	130	- 20	750	265
Acetal resin	D,W	Plastic chains	- 40	80	65	- 40	175	150
Low friction acetal resin	LF (all colors)	Plastic chains and belts	-40	80	65	-40	175	150
Anti static acetal resin	AS	Plastic chains and belts	-40	80	not applicable	-40	175	not applicable
NG®evo conveyor components made from engineered plastic resin	NG®evo	Plastic chains and belts	-40	120	60	-40	250	140
Polypropylene	PP	Plastic chains and belts	5	104	104	41	220	220
Chemical resistant	CR	Plastic chains	5	104	104	41	220	220
Polyamide - PA High temperature resistance	HT	Plastic chains and belts	0	140	not applicable	30	285	not applicable
Polyphenylene sulfide Extra high temperature resistance	XT	Plastic chains and belts	-40	240	not applicable	-40	464	not applicable
Polyamide - PA Abrasion resistance	AR	Plastic chains	0	90	90	30	195	195
Thermoplastic rubber SEBS	TPR	VG chains and belts, gripper chains	- 40	100	100	- 40	210	210
NBR rubber	NBR	Gripper chains	- 25	100	100	-15	210	210
EPDM rubber	EPDM	Gripper chains	-40	120	120	-40	250	250
EPDM-PP rubber	EPDM-PP	Gripper chains	-40	120	120	- 40	250	250
Polyamide	PA	Sprockets, components	0	90	90	30	195	195
Polyamide reinforced	PA FV	Sprockets, bearing, components	0	90	90	30	195	195
Polypropylene	PP	Components	5	104	104	41	220	220
Polypropylene reinforced	PP FV	Bearing, components	5	104	104	41	220	220
Polyethylene	PE	Curves, components	- 40	80	80	- 40	175	175
Nolu®-S parts	Nolu®-S parts	Curves, components	-40	80	80	-40	175	175



7 3 5 6 0 a Steel chain materials

7 3 5 7 3 a Nolu®-S parts

Storage of plastic chains and belts & RoHs Coefficient of friction of 7 3 5 7 8 a

standard materials

7 3 5 8 0 a Coefficient of friction - Measurement procedure 7 3 5 8 6 a standard materials Chemical resistance of





Chains & sprockets	Wear strips	Side guide brackets & accessories	Levelers	Engineering manual
Modular belts & sprockets	Chain & belt return systems	Frame & structure supports	Bearing supports	
Curves & tracks	Side guide solutions	Miscellaneous products	Equipment	



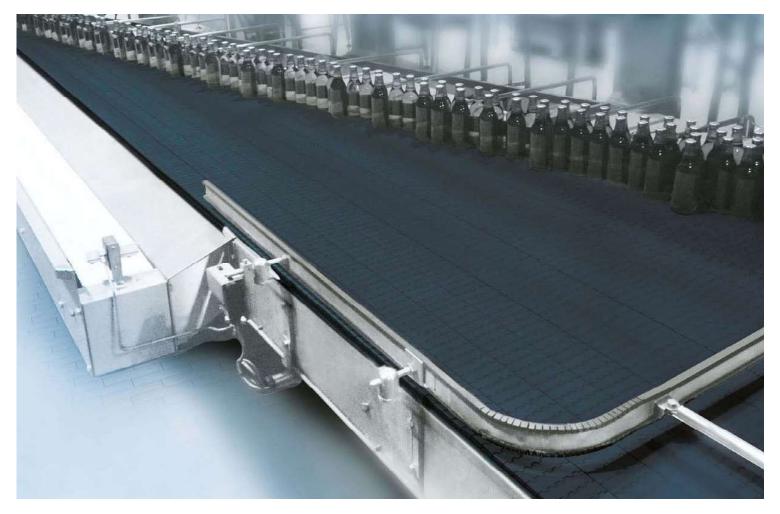
7 3 7 0 0 a Ng®evo conveyor components made from engineered plastic resin







Plastic Chains & Modular Belts









7 3 7 0 1 a

NG®EVO CONVEYOR COMPONENTS MADE FROM ENGINEERED PLASTIC RESIN







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WHAT PLASTIC MATERIALS ARE USED IN NEW GENERATION® CONVEYOR COMPONENTS?

The System Plast® New Generation® family of chains and belts conveyor components are made of proprietary engineered resins designed to provide a sustainable advantage over "industry standard" materials. Their reduced coefficient of friction properties enable end users to reduce or eliminate their chain/belt lubrication thus providing a true "dry running" conveyor. Better sliding properties also result in reduced power consumption, increased wear life, reduced dust generation and the ability to run at higher speeds.

SYSTEM PLAST PRODUCTS ARE LEADING THE WAY WITH INNOVATION!

We introduced the New Generation® brand of conveyor components 20 years ago. It was soon recognized to offer new possibilities for running conveyors lines without lubrication. This enabled the reduction/elimination of soap & water or dry lubricants creating a safer work environment and cost savings. NG® conveyor components have also proven they provide longer wear life in comparison to acetal materials, and reduce noise levels in dry applications.

Regal expertise with unique materials and in-house integrated tool development is critical for consistent control over design and manufacturing processes.

MATERIAL EVOLUTION TO MEET YOUR GOALS

Understanding the growing demands on safety, hygiene, sustainability and TCO (total cost of ownership) - Regal challenged itself to improve the original formulation. This successful development created a new and improved resin used to make NG® Evolution conveyor components which helps to meet your goals. Your floors can be drier, bacteria growth can be reduced, energy and water consumption can be less and your TCO will be improved.



IMPROVEMENTS OF NG® EVO COMPARED TO NG® CONVEYOR COMPONENTS

Lower friction - Friction is reduced up to 15%.

Higher strength - Up to 10%

Higher abrasion resistance

Approved for direct food contact according to EU and FDA regulations

MATERIALS AND CALCULATIONS - General index 7 3 5 1 0 a -







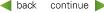
NG®EVO CONVEYOR COMPONENTS MADE FROM ENGINEERED PLASTIC RESIN

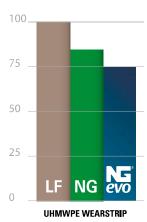












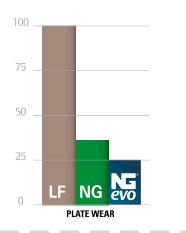
LOWEST COEFFICIENT OF FRICTION BETWEEN CHAIN /BELT AND WEARSTRIP

Coefficient of friction NG® evo conveyor components on UHMWPE wearstrip:

25% lower than LF acetal chain 15% lower than NG chain

Coefficient of friction values of 0,10 or even less are achievable in running dry applications

- Less power consumption
- Improved product stability
- Improved product flow
- Improved productivity



HIGHEST PLATE WEAR RESISTANCE

Plate wear in accelerated abrasion test after 5400 km run length

75% less wear than LF acetal chain 30% less wear than NG chain

Increased wear provides many advantages

- Less dust generation
- Reduced contamination
- Reduced cleaning requirements

NG® EVO CONVEYOR COMPONENTS ARE APPROVED FOR DIRECT FOOD CONTACT ACCORDING TO EU AND FDA **REGULATIONS**



NOISE REDUCTION:

With NG evo components the risk of noisy chains is greatly reduced compared to LF acetal chains. Squealing curves can be resolved and improve the work environment.



HIGH PV (PRESSURE VELOCITY) LIMIT:

With NG evo conveyor components, higher speeds and higher loads are achievable in dry running lines compared to LF Acetal chains. Additional speed and wear advantages can be also gained when used in combination with NOLU®-S or NOLU-SR curves.



CHEMICAL RESISTANCE:

NG evo conveyor components have a better chemical resistance than LF acetal materials, being compatible with most cleaning agents, especially in aseptic filling with the presence of H₂O₂ (hydrogen peroxide). The material in NG evo components, unlike many other materials used in the same application, do not get attacked by this chemical

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NG®EVO CONVEYOR COMPONENTS MADE FROM ENGINEERED PLASTIC RESIN











ENHANCED RUNNING DRY POSSIBILITIES WITH NG®EVO CONVEYOR COMPONENTS:

Contact Regal for assistance to help you achieve this goal.

An overview of the process is shown below:

STEP 1: GOAL TO RUN DRY

- Save water / lubricant consumption
- Eliminate wet floors safety
- Less bacteria growth hygiene
- Reduce maintenance
- Reduce energy consumption

STEP 2: DEFINE PROCESS PARAMETERS

- Layout
- Production / hour speeds
- Geometry bottle, can, etc.

STEP 3: DEVELOP A ROBUST PROCESS

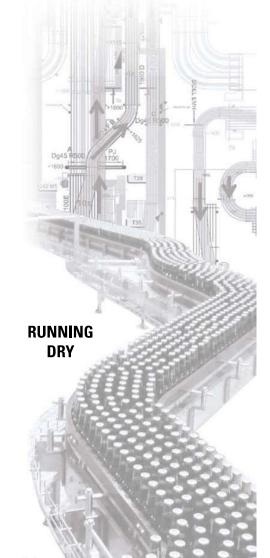
- Analyze the data from the previous step
- Discuss requirements and conditions with OEM and End user
- Select the correct productsi

STEP 4: IMPLEMENT DRY RUNNING

- Advise during installation
- Train operators

STEP 5: CONTROL AND IMPROVE PROCESS

- Monitor, follow up and make changes if necessary
- Analyze the collected data





MATERIALS AND CALCULATIONS - General index 7 3 5 1 0 a -



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NG®EVO CONVEYOR COMPONENTS MADE FROM ENGINEERED PLASTIC RESIN





A SUCCESS STORY OF NG® CONVEYOR COMPONENTS







DRY RUNNING SYSTEM PLAST® CONVEYOR COMPONENTS HELP HEINEKEN BREW A BETTER FUTURE WITH WATERLESS BOTTLING CONVEYORS IN ZOETERWOUDE PLANT

THE TWO WATER FOOTPRINT NETWORK PARTNERS TEAM UP TO IMPROVE PLANT SAFETY AND SUSTAINABILITY BY REDUCING WATER USE, ENERGY CONSUMPTION, AND SOUND LEVELS AT ZOETERWOUDE BREWERY.

Zoeterwoude, Netherlands 2014 - Heineken* N.V.s global sustainability strategy, known as "Brewing a Better Future," produced a company-wide reduction of 20 percent in water use between 2008 and 2013. Playing a role in that improvement are System Plast NG® conveyor chain/belts and NOLU®-S wear track from System Plast S.r.l., a subsidiary of Regal Beloit Corporation (referred to from here on as "Regal"). The ultra-low-friction components, which are being deployed in phases at Heineken's Zoeterwoude brewery, eliminate the need for water and chemical-based lubrication on the filling lines - including those where abrasive particles from aluminum cans, party kegs and returnable glass bottles have been a problem in the past.

According to managers at the brewery, the dry running conveyor is producing a cascade of sustainability improvements with a cleaner, quieter, more energy-efficient and reliable plant. "We have experienced a wide variety of gains from the System Plast dry running conveyor," said Mr. Cok Duivenvoorden, Technical Line Manager at Zoeterwoude. "Specifically, we have improved plant safety and hygiene with dry equipment and floors. Maintenance is easier because of the cleanliness and better access where drip trays have been removed. We have reduced costs for water, lubricating chemicals and wastewater discharge. Dry operation is easier on conveyor bearings and frames, yet still reduces energy consumption because of the low-friction components. System Plast NG chain lasts up to five times longer than

low-friction acetal in some of our applications. And when installed to replace worn-out conveyor, the new chain pays for itself in as little as a year."

A JOINT GOAL TO REDUCE WATER USE IN BEVERAGE PROCESSING

One goal of Heineken's "Brewing a Better Future" sustainability program is to reduce water consumption by at least 25% by 2020, and both Heineken and Regal are pursuing dry conveying solutions as members of the Water Footprint Network. The network is a global organization of businesses, governmental agencies and environmental groups that promotes the transition to sustainable use of fresh water resources through increased awareness of how production and consumption of goods affect fresh water systems.

*Heineken is believed to be a trademark and/or trade name of Heineken Brouwerijen B.V. and is not owned or controlled by Regal Beloit Corporation.

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APPLICATION CONSIDERATIONS



The proper selection and application of power transmission products and components, including the related area of product safety, is the responsibility of the customer. Operating and performance requirements and potential associated issues will vary appreciably depending upon the use and application of such products and components. The scope of the technical and application information included in this publication is necessarily limited. Unusual operating environments and conditions, lubrication requirements, loading supports, and other factors can materially affect the application and operating results of the products and components and the customer should carefully review its requirements. Any technical advice or review furnished by Regal Beloit America, Inc. and its affiliates with respect to the use of products and components is given in good faith and without charge, and Regal assumes no obligation or liability for the advice given, or results obtained, all such advice and review being given and accepted at customer's risk.

For a copy of our Standard Terms and Conditions of Sale, Disclaimers of Warranty, Limitation of Liability and Remedy, please contact Customer Service.

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2502 2508	30640a	882TW	11300a 11300b	CC-42-12MK CC-42-12MQ	53620a 53630a	GRPNBR GRPKNBR	11440a 11440a
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515RMS	11130a	BEO-19	56044a	CL	56090a	HDL-117	54560a
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800RMS	11120a	BK-55W43L	53535b	CL-CON-P25	51630a	ICOF-GLASS	57080a
800WMS	11120b	BKE-85W22	53555a	CL-CON-PD	51610a	ICOF-PET	57080a
800FSS	11120b	BKF-35W31L	53555a	CL-CON-R	51820a	ICOF-SLIDER-CAN	57080a
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815DS	11040a	BKF-66W31L	53515b	CL-H7816M	51690a	IND-350R	53635a
815FSS	11090d	BKF-79W30	53555a	CL-H8716M	51700a	MODFLEXA90R	57035a
815RMS	11040b	BKF-89W38	53560a	CL-H9014M	51680a	MODFLEXA180R	57055a
8157DMS	11100a	BKLSS	53580a	CL-RD10-P.,	51640a	MODFLEXA90L	57040a
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KMD82	50070a	LF1700	11530a	LFG2250FT-K330	30310a	LMP80 LMPH123	55705a
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KMD85	50101a	LF1702	11530a	LFG2250FT-PTVG		LSHOE-S330	50600b
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KMD87	50101a	LF1843 TAB	11470a	LFG2250FT-PTVG-3	30370d	LSS125	55675a
KMD90	50091a	LF820	11190a	LFG2250FT-PT2	30335b	LSS50	55675a
KMD91	50091a	LF820P	11190a	LFG2250FT-PT-K330	30310a	LSS75	55675a
KMD92	50091a	LF821	11320a	LFG2250FT-TAB	30710a	LSSC100	55690a
KMD93	50091a	LF8257	11310a	LFG2251FT	30440a	LSSC125	55690a
KMD96	50091a	LF828	11210a	LFG2251FTVG	30490a	LSSC75	55690a
KMD97	50101a	LF831	11190a	LFG2251FTVG-2	30490b	SNB2080	30029a
KNF-30H21	54610a	LF843	11460a	LFG2251FTVG-3	30490b	NB2120	30175a
KNF-40H26	54610a	LF845	11460a	LFG2251FTVGS	30495a	SNB2120	30175a
KNF-50H32	54610a	LF877TAB	11230a	LFG2251FTVGS-2	30495b	NGD1873TABVG	11420a
KNF-50H39	54600a	LF878TAB	11230a	LFG2251FTVGS-3	30495b	NGE1400	11590a
KNF-50H40	54600a	LF879	11240a	LFG2251FT-M	30700a	NGE1431	11590a
KNF-50H41	54600a	LF8790P	11240b	LFG2251FT-PVG	30490c	NGE1700	11530a
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